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BOUNDARY LAYER ON A CONTINUOUS FLAT PLATE
IN PARALLEL FLOW WITH SIMILAR AND NON-SIMILAR
BOUNDARY CONDITIONS

BY

VAUGHN H. M. FABER
B.S.M.A.E., Illinois Institute of Technology, 1980

THESIS

Submitted in partial fulfillment of the requirements
for the degree of Master of Science in Engineering
in the Graduate Studies Program
of the College of Engineering
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ABSTRACT

This thesis is a computational analysis of the incompressible laminar boundary layer on a continuous moving surface. The skin friction and heat transfer variations on a moving plate with similar and non-similar (constant) mass transfer at the wall are obtained for 210 cases for the similar boundary solution and 60 cases for the non-similar solution.

Both cases were solved numerically using the fourth order Runge-Kutta algorithm. For the non-similarity case, the local non-similarity method as described by Sparrow, Quack and Boerner was used.

The analysis shows that the shear stress for both the similar and non-similar cases is higher when the velocity of the wall is greater than the free stream velocity. It decays with increasing mass transfer at the wall. The shear stress also increases as the velocity difference is enlarged.

Analysis was also made to determine the heat transfer from the plate for the similar boundary layer cases. The results of the analysis show that heat transfer is greatest when the wall velocity equals the free stream velocity.

Also, heat transfer was enhanced by increasing the injection parameter and decays with increasing velocity difference.

Practical applications of this flow field are extrusion of porous plastic sheets, quenching of large perforated metallic sheets, and water condensing on aircraft during flight. Water condensation during flight is of interest in the problem of aircraft icing.

ACKNOWLEDGEMENTS

The preparation of this thesis required the assistance of many people. The number of people who gave me assistance, both great and small, was greatly appreciated; however, space does not permit listing all of them.

I would like to give special thanks to my advisor, Dr. Loren Anderson, whose many suggestions and guidance made this effort possible. I would also like to thank Padmanabha Chappidi for his valuable assistance and many helpful suggestions. Lastly, but by no means least, I would like to thank my wife, Loraine. Without her constant help and encouragement this thesis, much less graduate school, would not have been possible.

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LIST OF SYMBOLS

- C = dimensionless injection parameter for similar problem = $v_w(\nu U_r/x)^{-1/2}$
- f = function which is proportional to Ψ
- p = the derivative of f with respect to η
- q = the derivative of p with respect to η
- g = derivative of f with respect to ζ
- r = derivative of g with respect to η
- s = derivative of r with respect to η
- Nu = Nusselt number
- Pr = Prandtl number
- Re = Reynolds number
- t = derivative of θ with respect to η (in Runge-Kutta formulas)
- T = temperature
- u = local velocity parallel to the plate
- u = derivative of Φ with respect to η (in Runge-Kutta formulas)
- U = velocity parallel to the plate
- v = local velocity perpendicular to the plate
- $\Delta_n X_i$: $n = 1, 2, 3, 4$; $X = f, g, p, q, r, s, \theta, \Phi, t, u$: = Runge-Kutta difference values

- ζ = dimensionless injection parameter for
 non-similar problem = $(v_w/U_r)(2U_r x/\nu)^{1/2}$
- η = dimensionless coordinate perpendicular to plate
 = $y(U_r/\nu x)^{1/2}$ (similar case)
 = $y(U_r/2\nu x)^{1/2}$ (non-similar case)
- θ = dimensionless temperature = $(T - T_\infty)/(T_w - T_\infty)$
- ν = kinematic viscosity
- Φ = derivative of θ with respect to ζ
- Ψ = stream function
 = $(\nu U_r x)^{1/2} f(\eta)$ (similar case)
 = $(2\nu U_r x)^{1/2} f(\zeta, \eta)$ (non-similar case)

Subscripts

- i = index for cell indices
- r = reference
- w = wall condition
- x, y = differentiation with respect to x or y respectively
- ζ = differentiation with respect to ζ
- ∞ = free stream condition

INTRODUCTION

This thesis investigates the boundary layer on a moving continuous flat plate with similar and non-similar mass transfer. The first part of this thesis investigates the case of similar mass transfer and the second part deals with non-similar mass transfer. For purposes of this thesis, similar mass transfer is when the injection velocity at the wall is proportional to the reciprocal of the square root of x , where x is a characteristic dimension along the plate. Constant mass transfer is the non-similar case. For the similar and non-similar cases, velocity profiles are computed. For the similar case, temperature profiles are determined; however, in the non-similar case computational problems prevent temperature profiles from being obtained.

The continuous moving flat plate (CF) problem, shown in Figure 1b, is different from the Blasius flat plate problem (BF), shown in Figure 1a, as stated by Abdelhafez [1]. In the continuous moving flat plate (CF) problem the coordinate system is fixed in space, whereas in the Blasius flat plate (BF) problem the coordinate system is fixed on the plate. This relationship remains the same for the continuous moving flat plate (CF) problem and Blasius flat plate problem when boundary layer injection or suction is included.

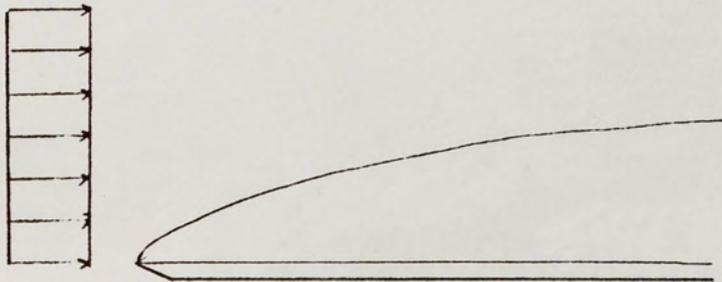


Figure 1a. Boundary Layer for Flow on a Flat Plate.

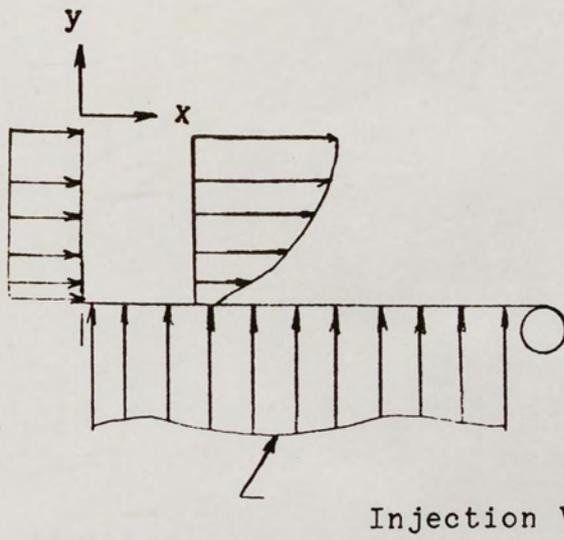


Figure 1b. Boundary Layer for Flow on a Moving Wall.

In the continuous moving flat plate (CF) problem, inclusion of blowing or suction at the plate does not affect the tendency for both the skin friction and heat transfer to be greater for cases where $U_w > U_\infty$ with the same velocity difference, $|U_w - U_\infty|$, and surface mass transfer. This is due to the change in direction of the transverse velocity component [1]. The differing values for skin friction and heat transfer between the continuous moving flat plate (CF) problem with injection or suction (CFI) and the Blasius flat plate (BF) problem with injection or suction (BFI) is an example that the two flows are different and cannot be mathematically transformed into one another with $|U_w - U_\infty|$ as a parameter using a Galilean transformation. A Galilean transformation is when components of a given vector in two different coordinate systems are related to each other by an orthogonal transformation [2].

ANALYSIS

The first part of this thesis explores the boundary layer for the CF problem with similar injection (the injection velocity $v_w = C(\nu U_r/x)^{1/2}$). The CFI problem is steady for a coordinate system fixed in space.

The second part of this thesis attempts to solve the CF problem with constant boundary layer injection at the plate surface. This problem is also known as non-similar boundary layer injection.

Similar Boundary Layer Case

As is usual for problems of this type, dimensionless variables are employed to make the results fit a broader range of applications and make some simplifications in the problem.

The following boundary layer equations can be derived by use of a rectangular control volume (subscripts denote differentiation with respect to the indicated variable):

$$u_x + v_y = 0 \quad (\text{continuity}) \quad (1)$$

$$uu_x + vu_y = \nu u_{yy} \quad (\text{momentum}) \quad (2)$$

and

$$uT_x + vT_y = (T_{yy})/Pr \quad (\text{energy}) \quad (3)$$

with the boundary conditions

$$\text{at } y = 0; u = U_w, v = v_w, T = T_w$$

where

$$v_w = C(\nu U_r/x)^{1/2}.$$

For this problem a dimensionless variable, η , is defined as

$$\eta = y(U_r/\nu x)^{1/2}.$$

From elementary fluid mechanics it is known that the stream function, Ψ , satisfies the continuity equation. The stream function for this problem, as in the BF problem, is defined as

$$\Psi = (U_r \nu x)^{1/2} f(\eta)$$

where U_r is the reference velocity and is defined as

$$U_r = U_\infty \text{ if } U_\infty > U_w$$

$$U_r = U_w \text{ if } U_w > U_\infty.$$

Also for a potential flow it can be shown that

$$u = \Psi_y \text{ and } v = -\Psi_x.$$

The solution to equation (2) with these variables is given by Schlichting [3] as

$$f'''' + (1/2)ff'' = 0 \tag{4}$$

and the transformed boundary conditions become

$$\text{at } \eta = 0; f = -2C \tag{5a}$$

$$f' = U_w/U_r \tag{5b}$$

$$\text{at } \eta = \infty; f' = U_\infty/U_r. \tag{5c}$$

Next, to solve equation (3), a dimensionless temperature, θ , is defined as

$$\theta = (T - T_\infty)/(T_w - T_\infty).$$

According to Burmeister [4], this transforms equation (3) to

$$\theta'' + (1/2)Prf\theta' = 0. \quad (6)$$

The boundary conditions become

$$\theta(0) = 1.0 \quad (7a)$$

$$\theta(\infty) = 0.0. \quad (7b)$$

Equation (4) with (5a), (5b) and (5c) are then solved numerically by the fourth order Runge-Kutta method.

Chow [5] gives, for solving equation (4), the equations to be used in the fourth order Runge-Kutta methods as

$$f' = p \quad (8a)$$

$$f'' = p' = q \quad (8b)$$

$$f''' = q' = -(1/2)fq. \quad (8c)$$

After applying the standard Runge-Kutta formulas, the following difference equations are obtained:

$$\Delta_1 f_i = hp_i \quad (9a)$$

$$\Delta_1 p_i = hq_i \quad (9b)$$

$$\Delta_1 q_i = -.5hf_i q_i \quad (9c)$$

$$\Delta_2 f_i = h(p_i + .5\Delta_1 p_i) \quad (9d)$$

$$\Delta_2 p_i = h(q_i + .5\Delta_1 q_i) \quad (9e)$$

$$\Delta_2 q_i = -.5h(f_i + .5\Delta_1 f_i)(q_i + .5\Delta_1 q_i) \quad (9f)$$

$$\Delta_3 f_i = h(p_i + .5\Delta_2 p_i) \quad (9g)$$

$$\Delta_3 p_i = h(q_i + .5\Delta_2 q_i) \quad (9h)$$

$$\Delta_3 q_i = -.5h(f_i + .5\Delta_2 f_i)(q_i + .5\Delta_2 q_i) \quad (9i)$$

$$\Delta_4 f_i = h(p_i + \Delta_3 p_i) \quad (9j)$$

$$\Delta_4 p_i = h(q_i + \Delta_3 q_i) \quad (9k)$$

$$\Delta_4 q_i = -.5h(f_i + \Delta_3 f_i)(q_i + \Delta_3 q_i). \quad (91)$$

These values are then used to compute f , p , and q at $i + 1$ as follows:

$$f_{i+1} = f_i + (\Delta_1 f_i + 2(\Delta_2 f_i + \Delta_3 f_i) + \Delta_4 f_i)/6 \quad (10a)$$

$$p_{i+1} = p_i + (\Delta_1 p_i + 2(\Delta_2 p_i + \Delta_3 p_i) + \Delta_4 p_i)/6 \quad (10b)$$

$$q_{i+1} = q_i + (\Delta_1 q_i + 2(\Delta_2 q_i + \Delta_3 q_i) + \Delta_4 q_i)/6. \quad (10c)$$

Since a boundary condition for $f''(0)$ is not given, a value is assumed for $f''(0)$ and the integration is carried out until $etamax$ is reached. The value of $f'(etamax)$ is compared to the value given for $f'(\infty)$ in equation (5c). For this problem $etamax$ is 10. If $f'(etamax)$ is within ϵ of $f'(\infty)$, then the assumed value of $f''(0)$ is taken as correct and the integration is stopped. If $f'(etamax)$ is not within ϵ of $f'(\infty)$, then a new value of $f''(0)$ is assumed. In order to make more efficient use of the computer, a new value for $f''(0)$ is chosen by use of the secant method. Zucrow [6] explains the secant method which is a numerical technique that significantly accelerates convergence in iterative calculations.

Once the solution of equation (4) is obtained, the solution to equation (6) can be solved. For equation (6) a different method is taken. Chow [5] derives a set of difference equations which are solved simultaneously to obtain the solution to a general second order ordinary differential equation. This method is executed in lines 77

through 100 and subroutine TRID in the program listing in Appendix A, with a detailed description given by Chow [5].

The result of this analysis is shown in Figures 2 - 16. Figures 2 - 5, 7 - 10 and 12 - 15 show the velocity and temperature profiles for various values of the similarity parameter (C) and velocity differences $|U_w - U_\infty|$. From these figures it is seen that the velocity profiles for the cases where $U_\infty > U_w$ are similar to those with $U_w > U_\infty$ for the same injection parameter (though rotated).

As shown in Figures 2 - 5, the boundary layer thickens as the injection parameter (C) is increased and as the velocity difference is increased. The reason for this behavior is that energy is added with increasing injection parameter (C) and this results in a more gradual velocity profile. Additionally, as the velocity difference between the free stream and plate increases, there is a greater velocity gap to span, thus resulting in a thicker boundary layer.

Figure 6 shows how the friction coefficient varies with both $U_w - U_\infty$ and the injection parameter (C). Figure 6 also shows that the friction coefficient (C_f), is greater when $U_w > U_\infty$ for comparable values of C than when $U_\infty > U_w$. This is, as stated earlier, the result of the change in direction of the transverse velocity vector.

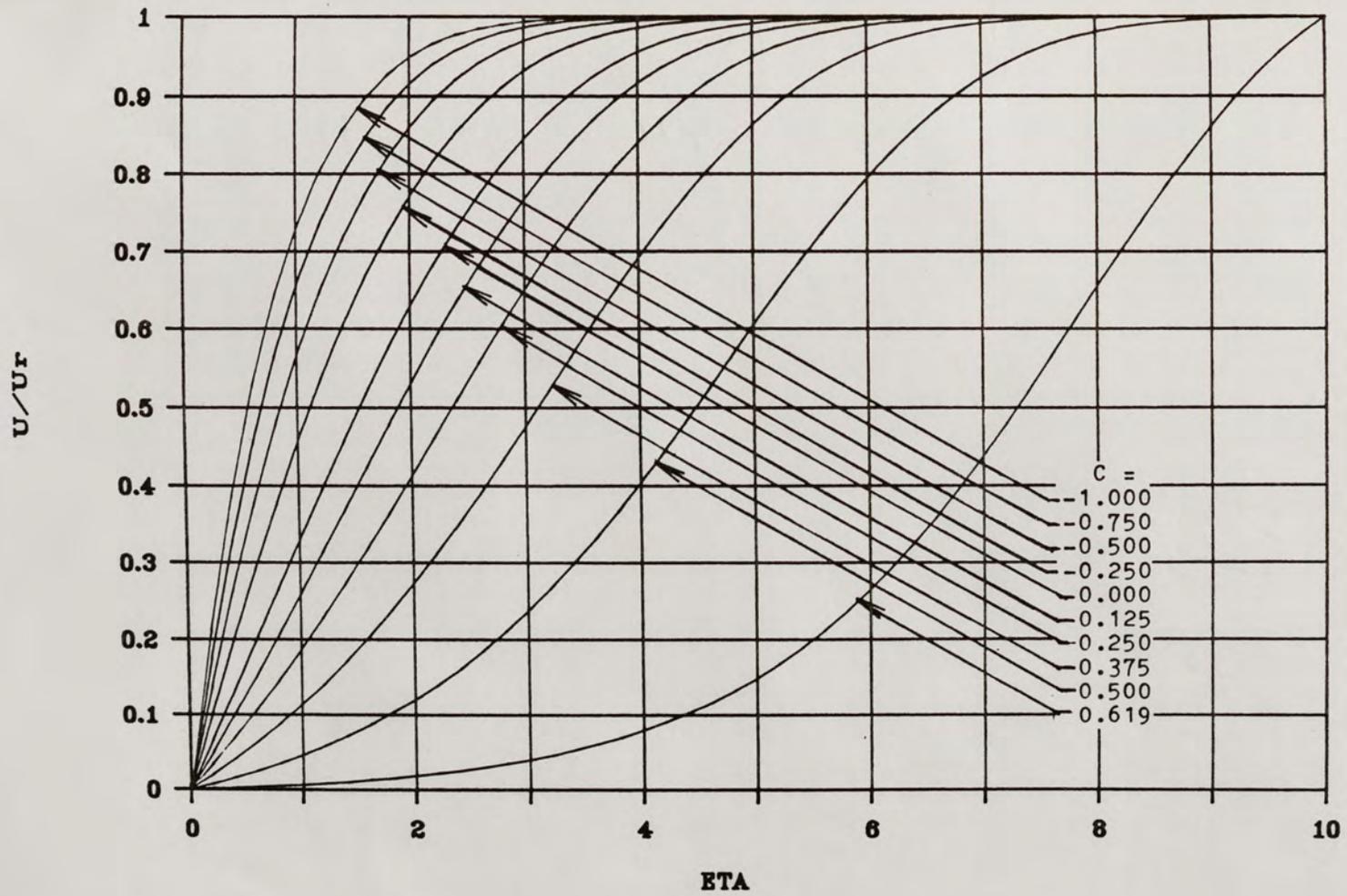


Figure 2. Velocity Profiles for $U_\infty - U_w = 1$.

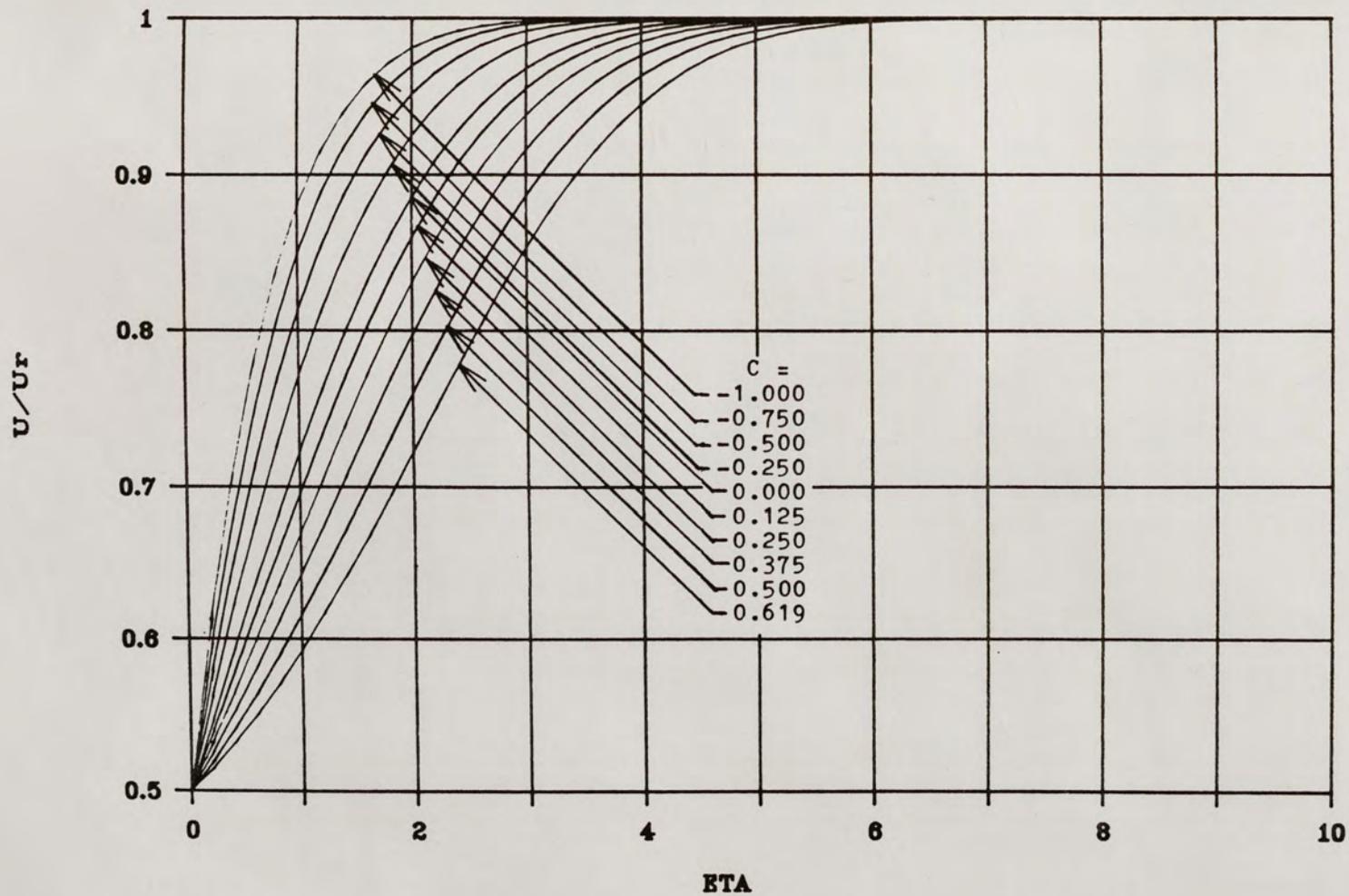


Figure 3. Velocity Profiles for $U_{\infty} - U_w = 0.5$.

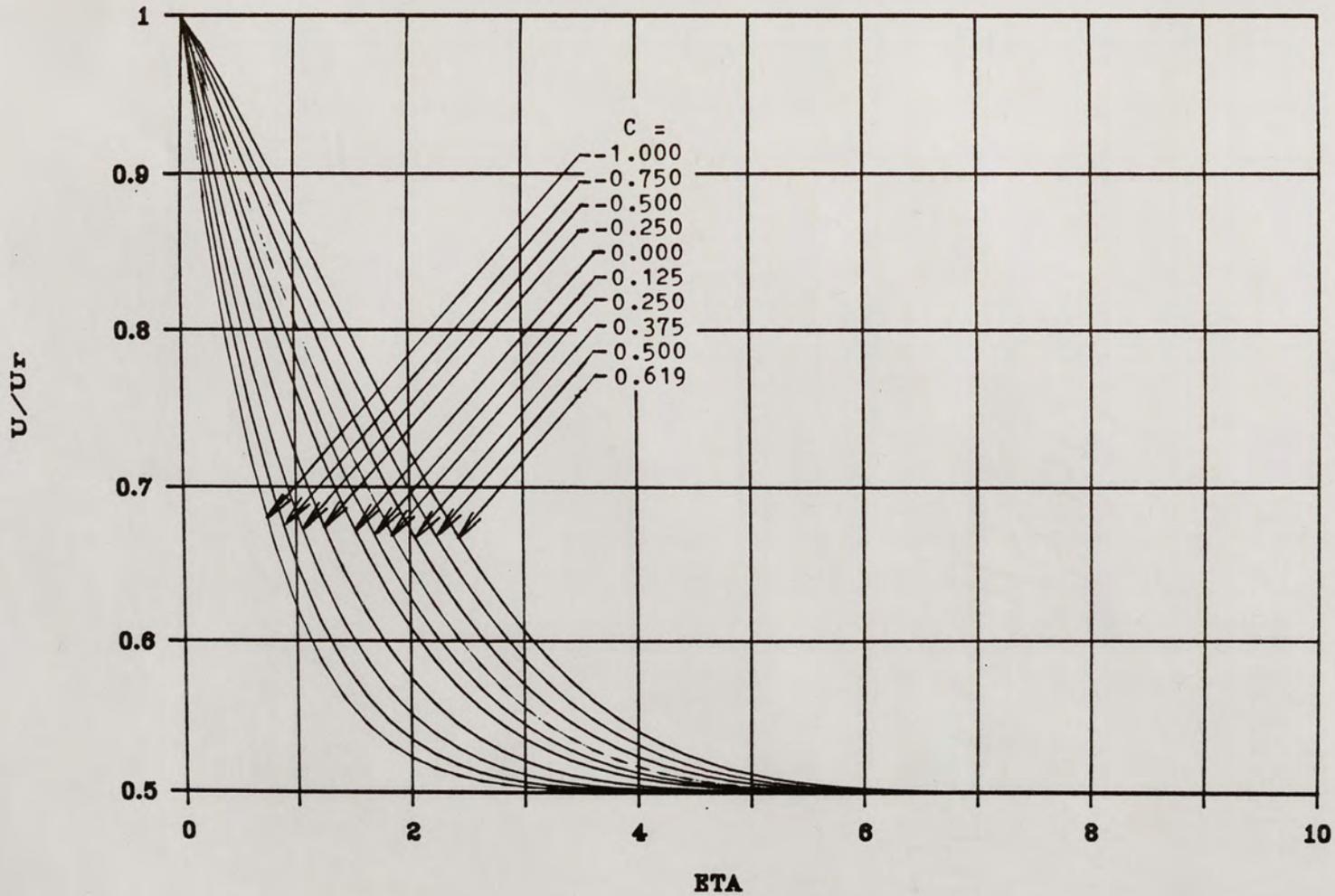


Figure 4. Velocity Profiles for $U_w - U_\infty = 0.5$.

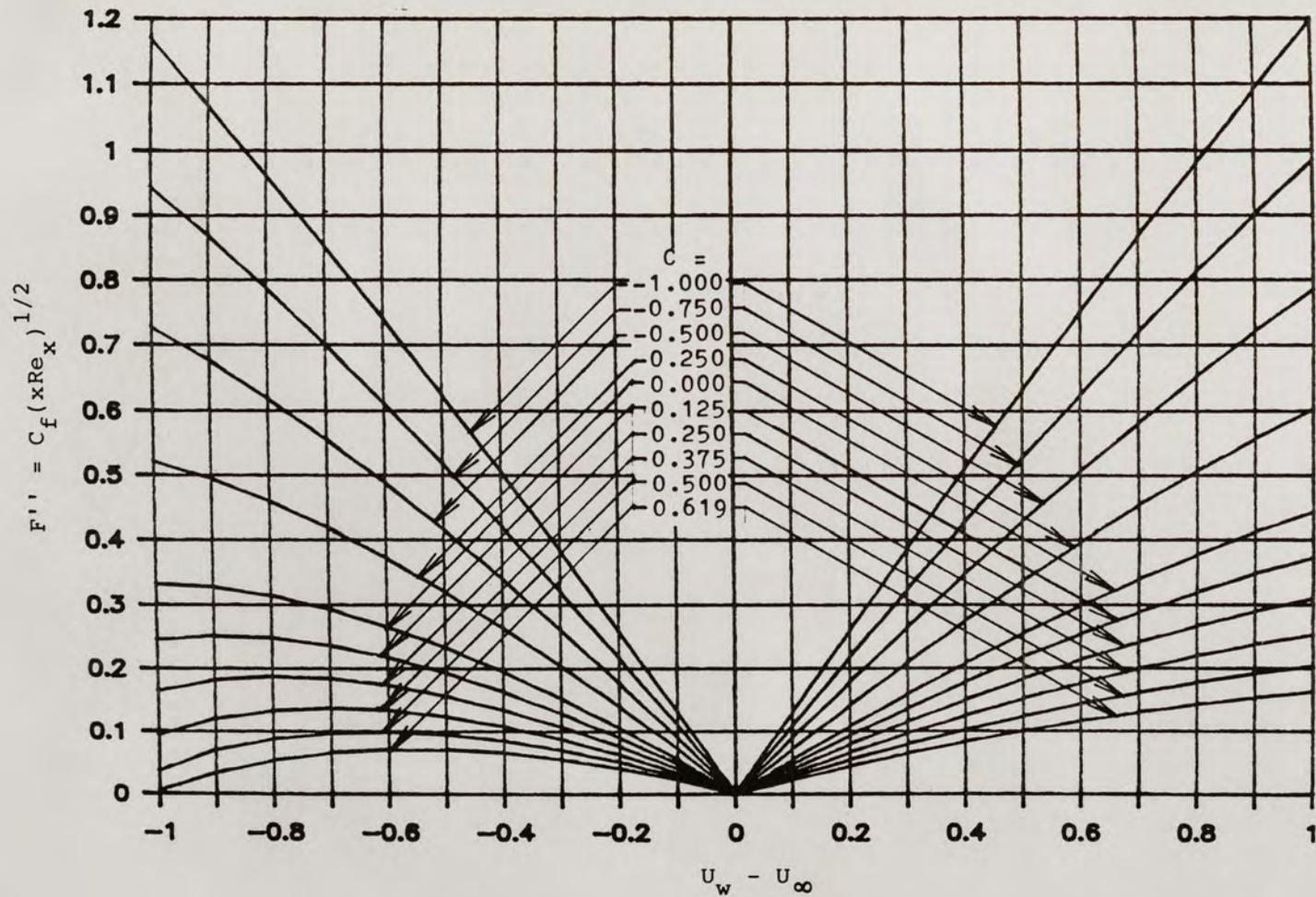


Figure 6. Friction Coefficient vs. Injection Parameter and Velocity Difference.

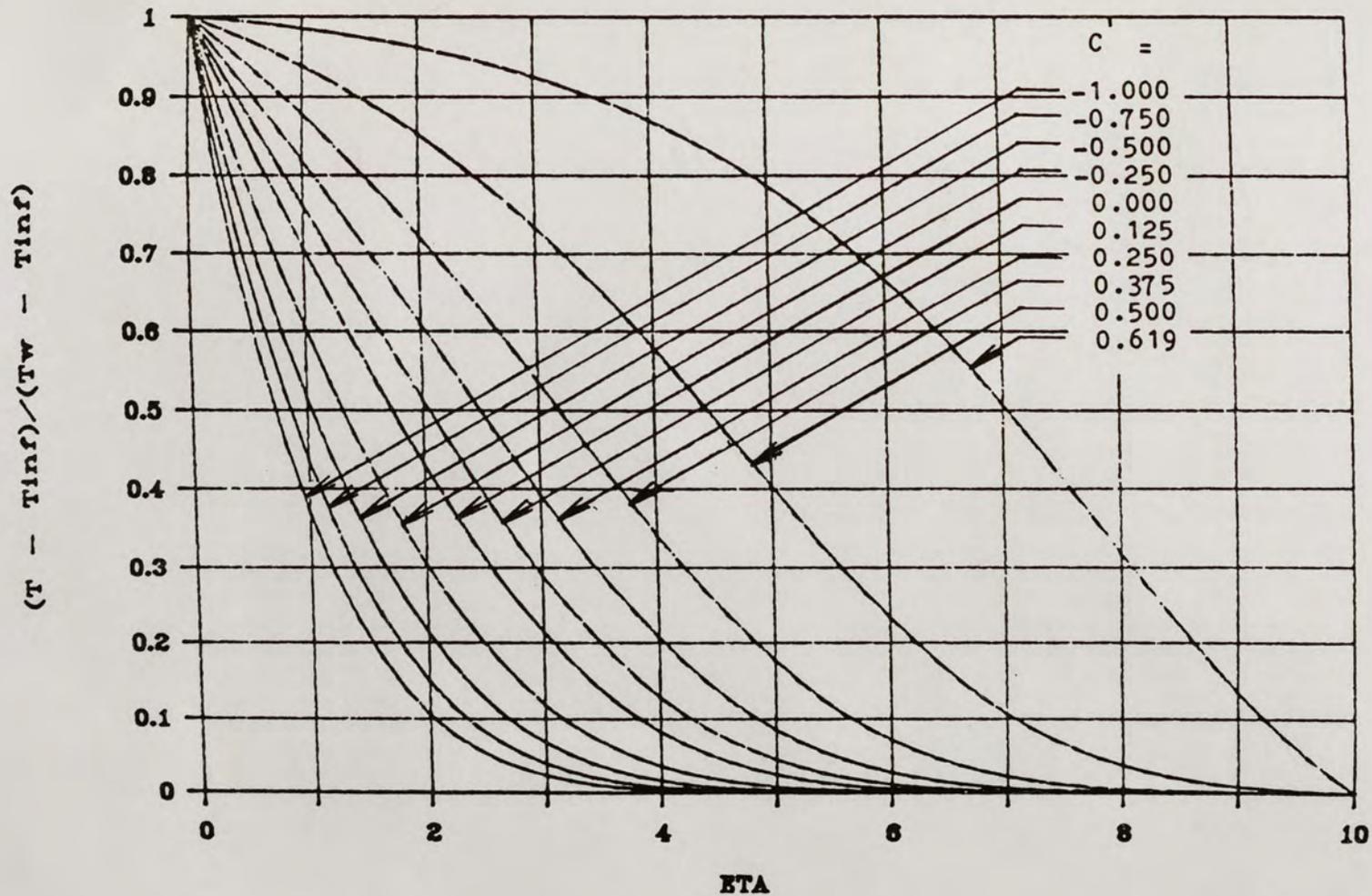


Figure 7. Temperature Profiles for $U_\infty - U_w = 1$ and $Pr = 0.72$.

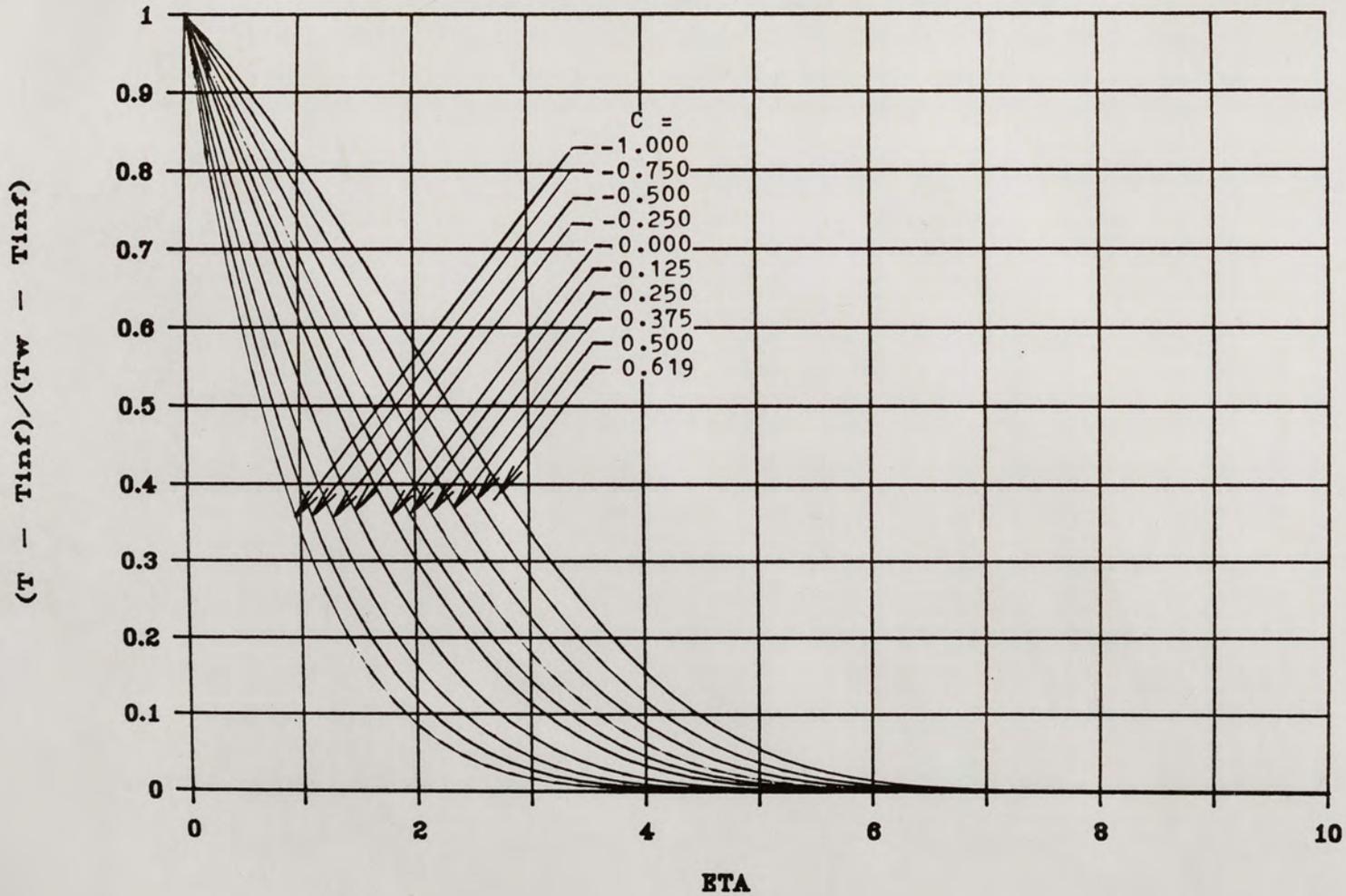


Figure 8. Temperature Profiles for $U_{\infty} - U_w = 0.5$ and $Pr = 0.72$.

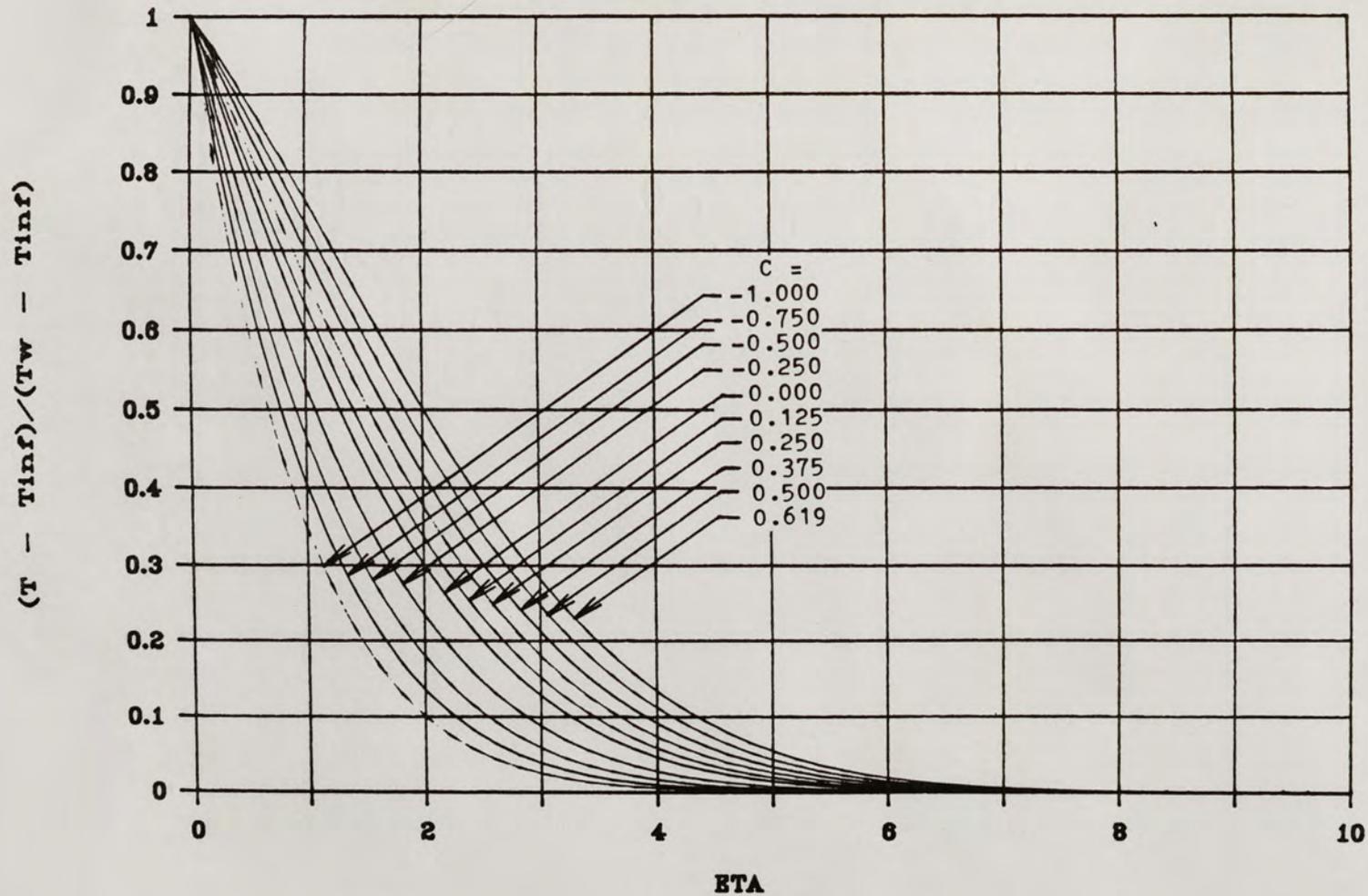


Figure 9. Temperature Profiles for $U_w - U_\infty = 0.5$ and $Pr = 0.72$.

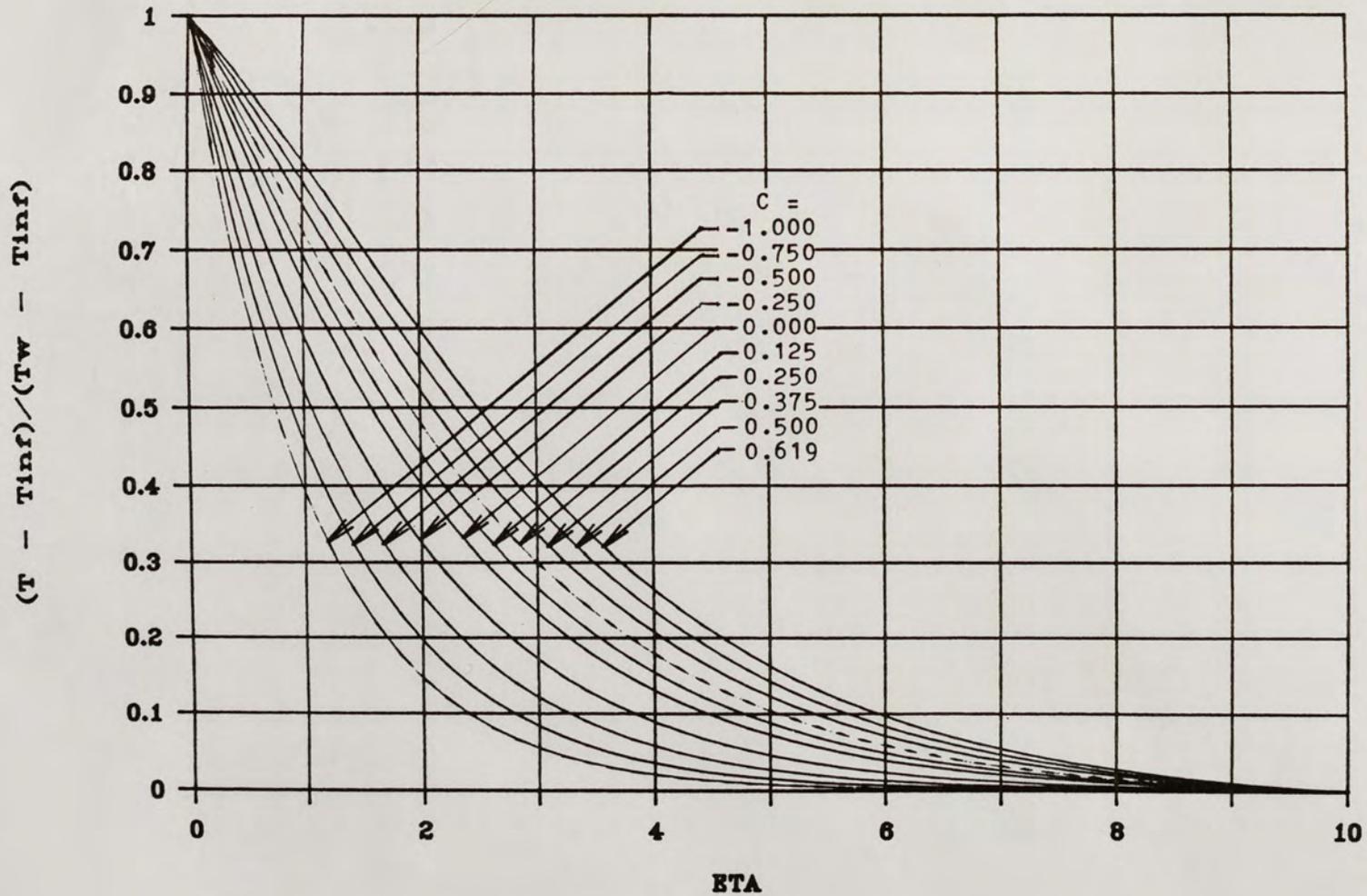


Figure 10. Temperature Profiles for $U_w - U_\infty = 1$ and $Pr = 0.72$.

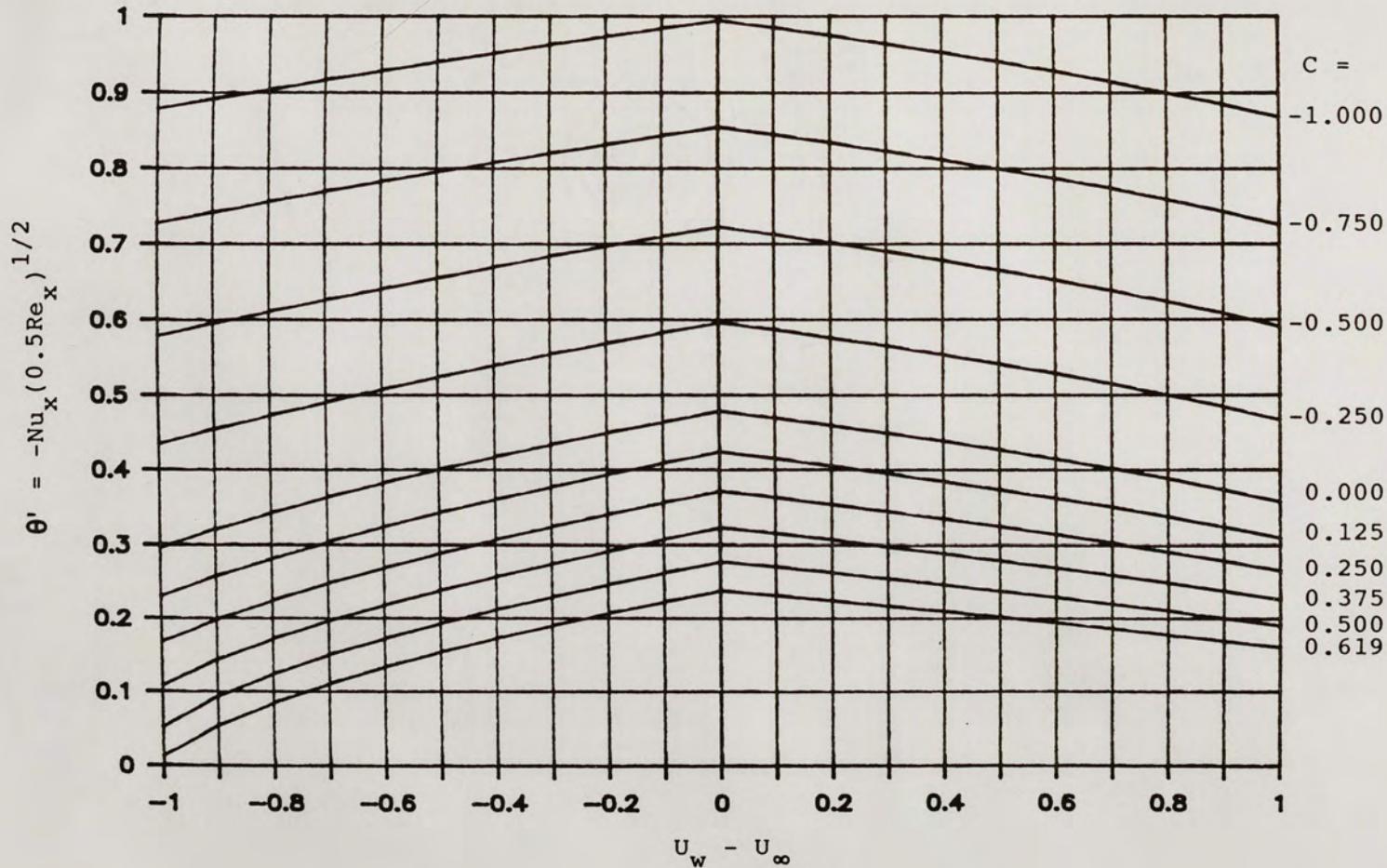


Figure 11. Nusselt Number vs. Injection Parameter and Velocity Difference for $Pr = 0.72$.

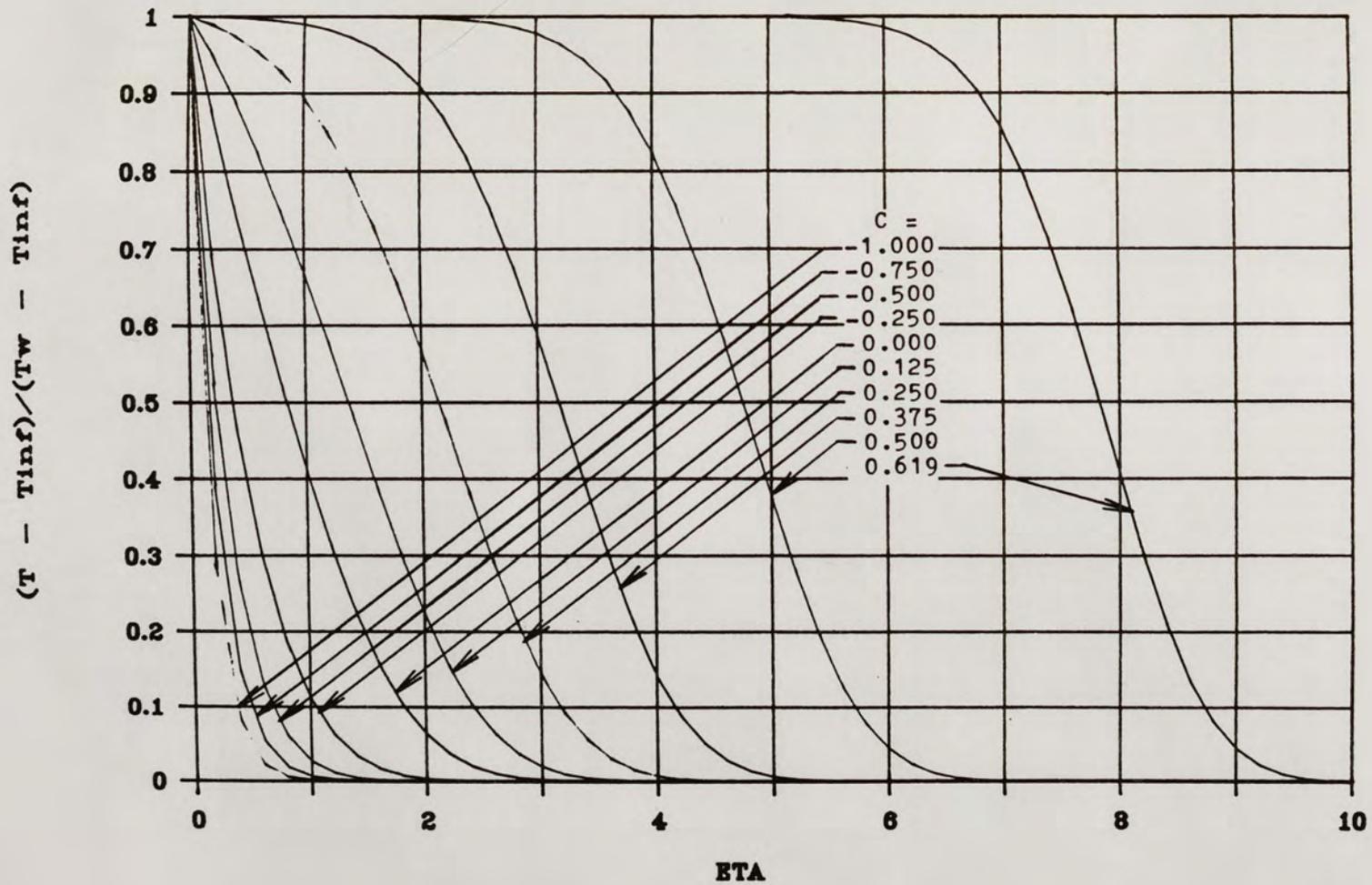


Figure 12. Temperature Profiles for $U_\infty - U_w = 1$ and $Pr = 6.0$.

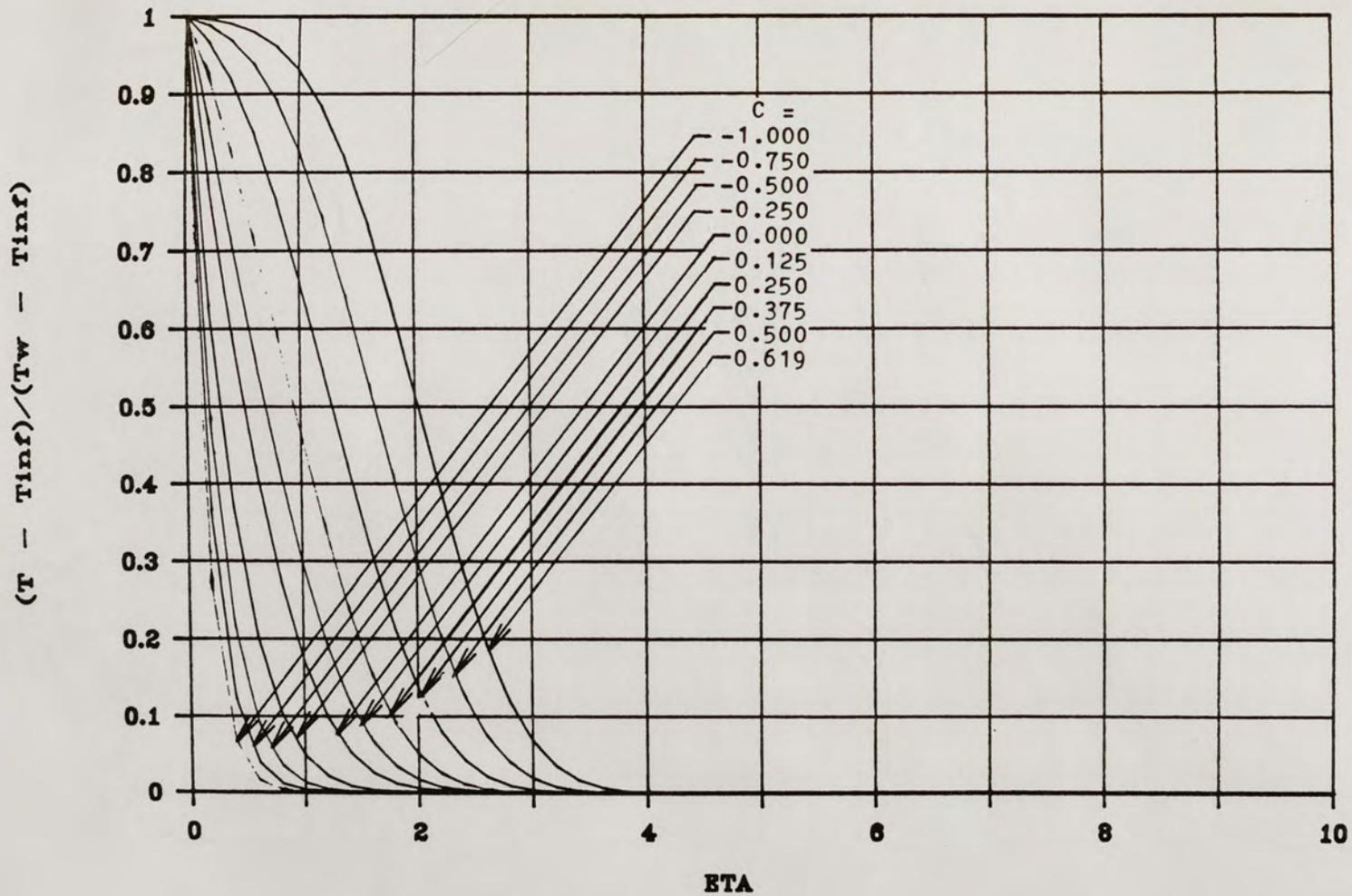


Figure 13. Temperature Profiles for $U_{\infty} - U_w = 0.5$ and $Pr = 6.0$.

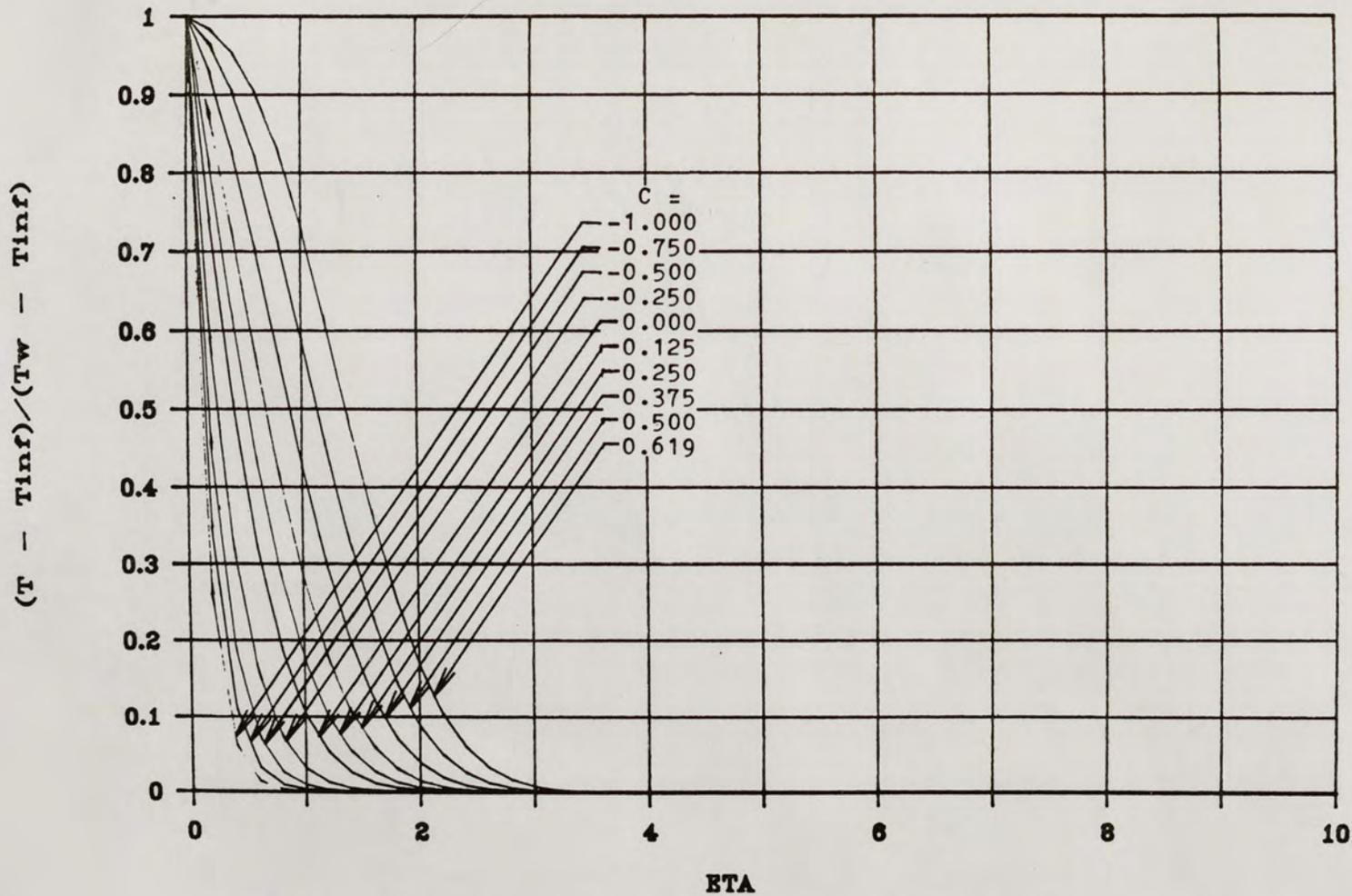


Figure 14. Temperature Profiles for $U_w - U_\infty = 0.5$ and $Pr = 6.0$.

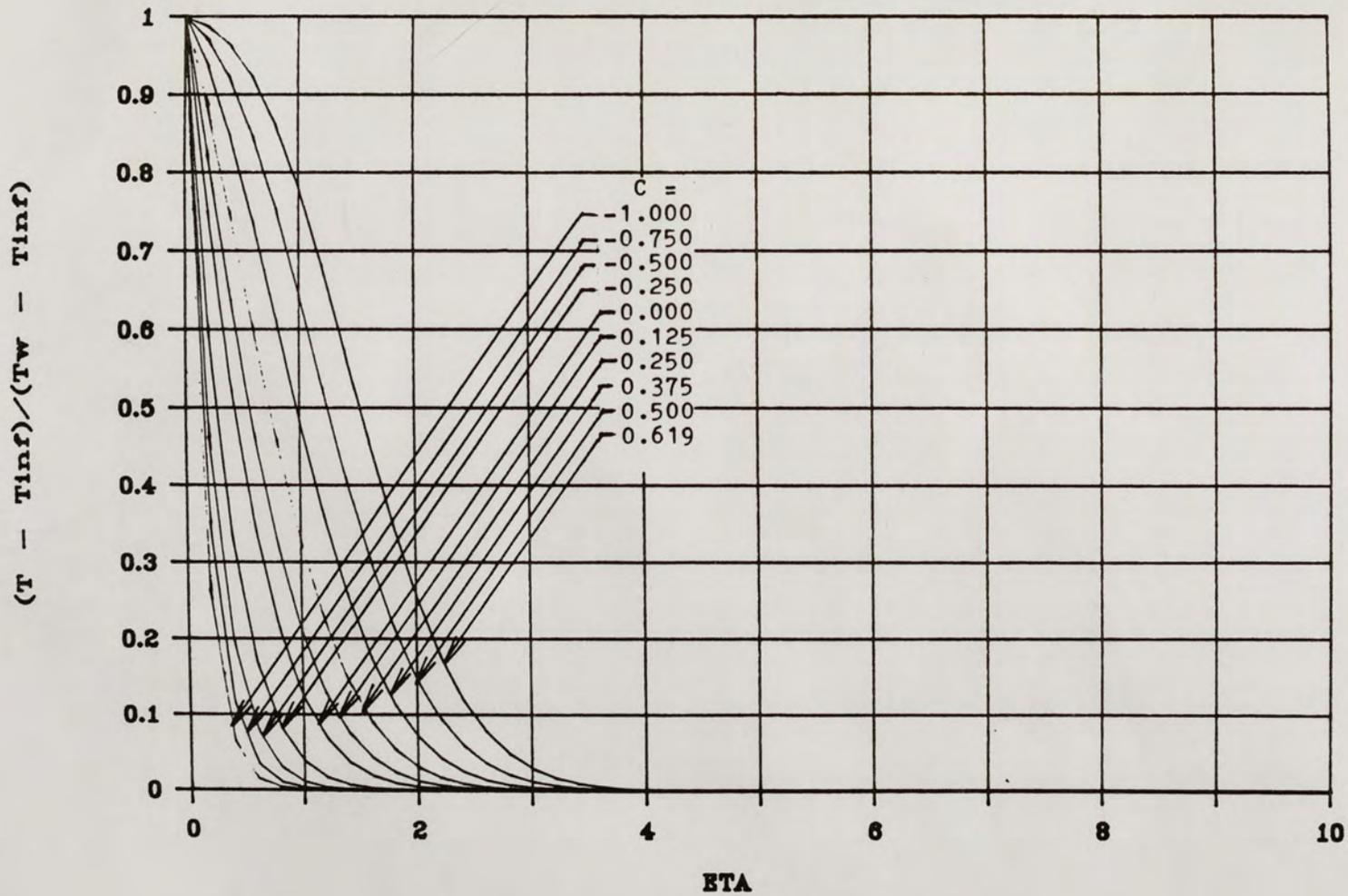


Figure 15. Temperature Profiles for $U_w - U_\infty = 1$ and $Pr = 6.0$.

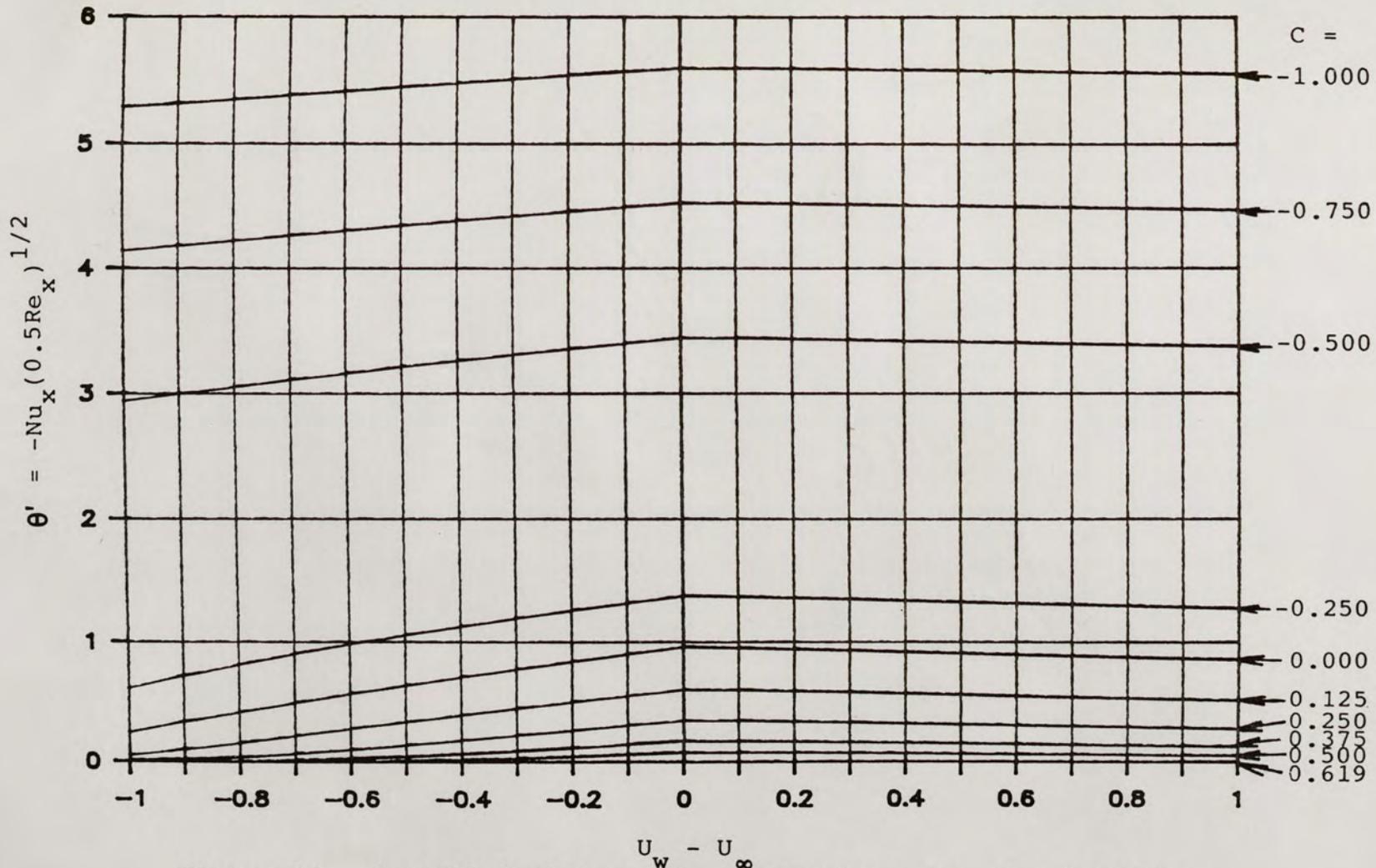


Figure 16. Nusselt Number vs. Injection Parameter and Velocity Differences for Pr = 6.0.

Figures 7 - 10 and 12 - 15 show the thermal boundary layer profiles for cases where $|U_\infty - U_w| = 1$ and 0.5. As with the momentum boundary layer, the thermal boundary layer is thicker for increasing similar injection parameter (C) and velocity difference. By comparing Figures 7 - 10 with 12 - 15, it is seen, as is the case for the thermal boundary layer in the classical Blasius problem, that the thermal boundary layer thickness increases with the Prandtl number, Pr. Figures 11 and 16 show how the Nusselt Number (Nu) varies with velocity difference ($U_w - U_\infty$), injection parameter (C), and Prandtl number (Pr). The most obvious thing that is seen when comparing Figures 11 and 16 is that the Nusselt number varies from about 3 to 6 times greater for Pr = 6 than for Pr = 0.72 for comparable cases. The next thing that is noticed is that the Nusselt number (Nu) increases with decreasing injection parameter (C), velocity difference $|U_w - U_\infty|$, and increasing Prandtl number (Pr). The increased Nu with increasing Pr is seen in equation (6). Tables 1 - 3 give the data from Figures 6, 11 and 16. Comparing data from Tables 1 and 2 with Figures 2 and 3 from Abdelhafez [1] show good agreement.

TABLE 1

FRICTION COEFFICIENT VS. VELOCITY DIFFERENCE AND INJECTION PARAMETER

$$(0.5C_f(Re_x)^{1/2}) \text{ VS. } U_w - U_\infty \text{ AND } C$$

$\frac{U_w - U_\infty}{U_\infty}$	Values of C									
	-1.0000	-0.7500	-0.5000	-0.2500	0.0000	0.1250	0.2500	0.3750	0.5000	0.6190
0.00	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.10	0.1309	0.1102	0.0905	0.0722	0.0554	0.0477	0.0406	0.0340	0.0281	0.0231
-0.10	0.1306	0.1098	0.0900	0.0715	0.0546	0.0469	0.0397	0.0330	0.0271	0.0220
0.20	0.2597	0.2182	0.1788	0.1421	0.1086	0.0934	0.0793	0.0663	0.0546	0.0447
-0.20	0.2584	0.2166	0.1767	0.1394	0.1054	0.0898	0.0754	0.0622	0.0503	0.0403
0.30	0.3862	0.3239	0.2647	0.2096	0.1597	0.1370	0.1160	0.0968	0.0795	0.0649
-0.30	0.3834	0.3202	0.2600	0.2036	0.1522	0.1287	0.1070	0.0872	0.0695	0.0547
0.40	0.5104	0.4271	0.3482	0.2748	0.2084	0.1784	0.1506	0.1254	0.1027	0.0836
-0.40	0.5054	0.4206	0.3397	0.2638	0.1946	0.1631	0.1341	0.1077	0.0842	0.0648
0.50	0.6321	0.5278	0.4290	0.3373	0.2548	0.2175	0.1832	0.1521	0.1242	0.1009
-0.50	0.6244	0.5176	0.4155	0.3198	0.2325	0.1928	0.1563	0.1233	0.0942	0.0704
0.60	0.7513	0.6258	0.5070	0.3972	0.2985	0.2542	0.2135	0.1768	0.1440	0.1166
-0.60	0.7402	0.6110	0.4873	0.3712	0.2652	0.2171	0.1730	0.1334	0.0988	0.0710
0.70	0.8677	0.7208	0.5821	0.4541	0.3396	0.2884	0.2416	0.1994	0.1620	0.1308
-0.70	0.8527	0.7007	0.5549	0.4177	0.2924	0.2356	0.1836	0.1372	0.0972	0.0658
0.80	0.9812	0.8128	0.6540	0.5078	0.3777	0.3198	0.2671	0.2198	0.1781	0.1435
-0.80	0.9619	0.7865	0.6179	0.4589	0.3134	0.2474	0.1871	0.1337	0.0884	0.0540
0.90	1.0915	0.9013	0.7222	0.5580	0.4126	0.3483	0.2900	0.2380	0.1922	0.1545
-0.90	1.0675	0.8681	0.6760	0.4942	0.3270	0.2511	0.1817	0.1207	0.0700	0.0339
1.00	1.1982	0.9860	0.7864	0.6041	0.4439	0.3735	0.3100	0.2536	0.2044	0.1639
-1.00	1.1694	0.9454	0.7289	0.5228	0.3321	0.2447	0.1645	0.0936	0.0356	0.0046

TABLE 2

NUSSELT NUMBER VS. INJECTION PARAMETER AND VELOCITY DIFFERENCE FOR AIR

$$(\theta' = -Nu_x (Re_x)^{-1/2} \text{ FOR } Pr = 0.72)$$

$\frac{U_w - U_\infty}{U_\infty}$	Values of C									
	-1.0000	-0.7500	-0.5000	-0.2500	0.0000	0.1250	0.2500	0.3750	0.5000	0.6190
0.00	0.9946	0.8558	0.7225	0.5962	0.4787	0.4239	0.3720	0.3233	0.2780	0.2382
0.10	0.9843	0.8452	0.7119	0.5858	0.4688	0.4143	0.3629	0.3148	0.2701	0.2310
-0.10	0.9841	0.8443	0.7100	0.5827	0.4643	0.4091	0.3570	0.3080	0.2627	0.2230
0.20	0.9736	0.8343	0.7009	0.5750	0.4585	0.4045	0.3536	0.3061	0.2621	0.2238
-0.20	0.9733	0.8326	0.6972	0.5688	0.4494	0.3938	0.3412	0.2921	0.2467	0.2071
0.30	0.9626	0.8230	0.6895	0.5638	0.4480	0.3944	0.3441	0.2972	0.2540	0.2164
-0.30	0.9624	0.8206	0.6840	0.5544	0.4338	0.3777	0.3248	0.2754	0.2300	0.1905
0.40	0.9511	0.8112	0.6776	0.5522	0.4370	0.3839	0.3342	0.2881	0.2457	0.2089
-0.40	0.9512	0.8082	0.6704	0.5395	0.4176	0.3609	0.3075	0.2577	0.2121	0.1728
0.50	0.9392	0.7989	0.6652	0.5400	0.4256	0.3731	0.3240	0.2787	0.2372	0.2013
-0.50	0.9398	0.7956	0.6564	0.5239	0.4005	0.3431	0.2891	0.2389	0.1931	0.1539
0.60	0.9267	0.7860	0.6522	0.5273	0.4136	0.3617	0.3135	0.2690	0.2284	0.1935
-0.60	0.9281	0.7826	0.6418	0.5077	0.3825	0.3243	0.2695	0.2187	0.1726	0.1335
0.70	0.9136	0.7724	0.6384	0.5138	0.4010	0.3498	0.3024	0.2588	0.2193	0.1854
-0.70	0.9162	0.7692	0.6268	0.4907	0.3634	0.3040	0.2482	0.1966	0.1500	0.1109
0.80	0.8997	0.7579	0.6236	0.4993	0.3876	0.3372	0.2907	0.2483	0.2099	0.1772
-0.80	0.9039	0.7554	0.6111	0.4728	0.3428	0.2820	0.2248	0.1720	0.1244	0.0853
0.90	0.8848	0.7422	0.6076	0.4836	0.3732	0.3237	0.2784	0.2372	0.2002	0.1687
-0.90	0.8914	0.7411	0.5947	0.4537	0.3204	0.2577	0.1984	0.1433	0.0939	0.0541
1.00	0.8685	0.7249	0.5897	0.4662	0.3574	0.3092	0.2653	0.2256	0.1901	0.1601
-1.00	0.8785	0.7263	0.5775	0.4333	0.2956	0.2300	0.1670	0.1071	0.0511	0.0119

TABLE 3

NUSSELT NUMBER VS. INJECTION PARAMETER AND VELOCITY DIFFERENCE FOR WATER

$$(\theta' = -Nu_x (Re_x)^{-1/2} \text{ FOR } Pr = 6.0)$$

$\frac{U_w - U_\infty}{U_\infty}$	Values of C									
	-1.0000	-0.7500	-0.5000	-0.2500	0.0000	0.1250	0.2500	0.3750	0.5000	0.6190
0.00	5.5991	4.5340	3.4448	2.3684	1.3813	0.9598	0.6107	0.3478	0.1731	0.0771
0.10	5.5938	4.5275	3.4366	2.3583	1.3698	0.9483	0.6001	0.3392	0.1672	0.0736
-0.10	5.5700	4.4986	3.4014	2.3160	1.3228	0.9020	0.5582	0.3053	0.1436	0.0596
0.20	5.5885	4.5210	3.4285	2.3483	1.3583	0.9369	0.5898	0.3309	0.1614	0.0702
-0.20	5.5404	4.4625	3.3567	2.2616	1.2617	0.8420	0.5043	0.2629	0.1156	0.0440
0.30	5.5833	4.5146	3.4204	2.3383	1.3469	0.9256	0.5795	0.3226	0.1558	0.0670
-0.30	5.5104	4.4255	3.3107	2.2050	1.1977	0.7794	0.4491	0.2209	0.0893	0.0306
0.40	5.5782	4.5082	3.4125	2.3284	1.3357	0.9144	0.5694	0.3146	0.1504	0.0639
-0.40	5.4799	4.3877	3.2632	2.1459	1.1305	0.7140	0.3924	0.1796	0.0654	0.0195
0.50	5.5731	4.5019	3.4046	2.3187	1.3246	0.9036	0.5596	0.3069	0.1453	0.0610
-0.50	5.4489	4.3490	3.2141	2.0840	1.0595	0.6454	0.3345	0.1396	0.0444	0.0111
0.60	5.5681	4.4957	3.3969	2.3092	1.3138	0.8929	0.5501	0.2994	0.1403	0.0583
-0.60	5.4172	4.3093	3.1631	2.0188	0.9840	0.5731	0.2754	0.1018	0.0269	0.0052
0.70	5.5631	4.4896	3.3893	2.2999	1.3033	0.8826	0.5409	0.2922	0.1357	0.0557
-0.70	5.3850	4.2685	3.1101	1.9498	0.9031	0.4966	0.2158	0.0674	0.0137	0.0018
0.80	5.5582	4.4836	3.3819	2.2909	1.2932	0.8727	0.5322	0.2855	0.1313	0.0534
-0.80	5.3521	4.2265	3.0548	1.8762	0.8157	0.4154	0.1565	0.0380	0.0050	0.0004
0.90	5.5534	4.4778	3.3747	2.2822	1.2836	0.8634	0.5240	0.2793	0.1274	0.0513
-0.90	5.3186	4.1832	2.9968	1.7971	0.7199	0.3288	0.0996	0.0158	0.0009	0.0000
1.00	5.5487	4.4721	3.3678	2.2740	1.2747	0.8549	0.5166	0.2736	0.1238	0.0494
-1.00	5.2843	4.1385	2.9357	1.7111	0.6133	0.2366	0.0487	0.0031	0.0000	0.0000

Non-Similar Boundary Layer Case

The second part of this thesis describes the calculation of the momentum and thermal boundary layers for the moving flat plate with uniform mass transfer at the plate surface. This boundary condition is an example of a non-similar boundary condition. It is called non-similar because the boundary condition and governing equations cannot both be expressed in terms of a single variable or parameter.

Equations (1) - (3) are solved and repeated here for convenience.

$$u_x + v_y = 0 \quad (1)$$

$$uu_x + vu_y = \nu u_{yy} \quad (2)$$

and

$$uT_x + vT_y = (T_{yy})/Pr. \quad (3)$$

The boundary conditions for this flow field are given as

$$\text{at } y = 0, \eta = 0; u = U_w, v = v_w, T = T_w$$

$$\text{at } y = \infty, \eta = \infty; u = U_\infty$$

where in this case $v_w = \text{constant}$.

The solution is obtained by assuming a stream function as follows:

$$\Psi = (2\nu U_r x)^{1/2} f(\zeta, \eta)$$

where

$$U_r = U_\infty \quad \text{if } U_\infty > U_w$$

$$U_r = U_w \quad \text{if } U_w > U_\infty$$

(as above), and

$$\zeta = (v_w/U_r)(2U_r x/\nu)^{1/2}$$

$$\zeta = v_w/U_r (2Re)^{1/2}$$

$$\eta = y(U_r/2\nu x)^{1/2}.$$

It is known that the stream function, Ψ , satisfies the continuity equation. It is also known that $u = \Psi_y$ and $v = -\Psi_x$.

Performing the necessary differentiations to obtain u , u_x , u_y , u_{yy} and v yields:

$$u = U_r f' \quad (11)$$

$$u_x = (U_r/2x)(\zeta f' - \eta f'')$$

$$u_y = (U_r/2x)(2U_r x/\nu)^{1/2} f'' \quad (13)$$

$$u_{yy} = (U_r/2x)(U_r/\nu) f''' \quad (14)$$

$$v = -.5(2\nu U_r/x)^{1/2}(f + \zeta f'_\zeta - \eta f'). \quad (15)$$

Substituting equations (11) - (15) into equation (2) yields:

$$f''' + ff'' = \zeta(f'_\zeta f' - f'' f'_\zeta). \quad (16)$$

Note: Primes denote differentiation with respect to η .

From equation (11) and the boundary conditions, it follows that

$$f'(\zeta, 0) = U_w/U_r. \quad (17)$$

Substituting $\eta = 0$, into equation (15), we get

$$f'_\zeta + (1/\zeta)f = -1$$

which can be solved by use of an integrating factor to obtain

$$f(\zeta, 0) = -\zeta/2.$$

Finally, at $\eta = \infty$, $u = U_\infty$, taken with equation (11), the final boundary condition is found to be

$$f'(\zeta, \infty) = U_\infty/U_r. \quad (19)$$

If we let the partial derivative of f with respect to ζ be represented by g , rewriting equation (16), we get

$$f''' + ff'' = \zeta(f'g' - f''g). \quad (20)$$

Next, the local non-similarity method described in Sparrow, Quack and Boerner [7] is applied. This is achieved by differentiating equation (20) with respect to ζ . This result is

$$g''' + fg'' - f'g' + 2f''g = 0. \quad (21)$$

The boundary conditions for this equation are found to be

$$g(\zeta, 0) = -1/2; \quad g'(\zeta, 0) = 0; \quad g'(\zeta, \infty) = 0.$$

To solve the energy equation, a non-dimensional variable, Theta, θ , is defined as

$$\theta(\zeta, \eta) = (T - T_\infty)/(T_w - T_\infty). \quad (22)$$

This implies that $T = (T_w - T_\infty)\theta + T_\infty$, which yields:

$$T_x = (T_w - T_\infty)(1/2x)(\zeta\theta_\zeta - \eta\theta') \quad (23)$$

$$T_y = (T_w - T_\infty)\theta'(U_r/2\nu x)^{1/2} \quad (24)$$

$$T_{yy} = (T_w - T_\infty)\theta''(U_r/2\nu x). \quad (25)$$

Substituting equations (11) - (14) and (23) - (25) into equation (3) results in

$$(1/Pr)\theta'' + f\theta' = \zeta(f'\theta_\zeta - \theta'f)$$

or

$$(1/Pr)\theta'' + f\theta' = \zeta(f'\Phi - \theta'g) \quad (26)$$

where Φ is defined as

$$\Phi = \theta_\zeta.$$

The boundary conditions become

$$\theta(y=0, \eta=0; \theta(\zeta, 0) = 1$$

$$\theta(y=\infty, \eta=\infty; \theta(\zeta, \infty) = 0.$$

For the local non-similarity model differentiate equation (26) with respect to ζ to obtain

$$(1/Pr)\Phi'' + 2g\theta' + \Phi'f - f'\Phi = 0 \quad (27)$$

where

$$\theta(y=0, \eta=0; \Phi(\zeta, 0) = 0$$

$$\theta(y=\infty, \eta=\infty; \Phi(\zeta, \infty) = 0.$$

Equations (20), (21), (26) and (27) are then programmed for solution by means of the fourth order Runge-Kutta method. To solve equations (20) and (21) by the Runge-Kutta method, the following values were defined:

$$f' = p \quad (28a)$$

$$f'' = q \quad (28b)$$

$$f''' = \zeta(f'g' - f''g) - ff' \quad (28c)$$

$$g' = r \quad (28d)$$

$$g'' = s \quad (28e)$$

$$g''' = f'g' - fg'' - 2f''g \quad (28f)$$

Equations (28 a-f) are then used to generate the finite difference equations used in the fourth order Runge-Kutta method as follows:

$$\Delta_1 f_i = h p_i \quad (29a)$$

$$\Delta_1 p_i = h q_i \quad (29b)$$

$$\Delta_1 q_i = \zeta(p_i r_i - q_i g_i) - f_i p_i \quad (29c)$$

$$\Delta_1 g_i = h r_i \quad (29d)$$

$$\Delta_1 r_i = h s_i \quad (29e)$$

$$\Delta_1 s_i = p_i r_i - f_i s_i - 2q_i g_i \quad (29f)$$

$$\Delta_2 f_i = h(f_i + 0.5\Delta_1 p_i) \quad (29g)$$

$$\Delta_2 p_i = h(p_i + 0.5\Delta_1 q_i) \quad (29h)$$

$$\begin{aligned} \Delta_2 q_i = h\{ & \zeta[(p_i + 0.5\Delta_1 p_i)(r_i + 0.5\Delta_1 r_i) \\ & - (q_i + 0.5\Delta_1 q_i)(g_i + 0.5\Delta_1 g_i)] \\ & - (f_i + 0.5\Delta_1 f_i)(p_i + 0.5\Delta_1 p_i)\} \end{aligned} \quad (29i)$$

$$\Delta_2 g_i = h(r_i + 0.5\Delta_1 r_i) \quad (29j)$$

$$\Delta_2 r_i = h(s_i + 0.5\Delta_1 s_i) \quad (29k)$$

$$\begin{aligned} \Delta_2 s_i = h[& (p_i + 0.5\Delta_1 p_i)(r_i + 0.5\Delta_1 r_i) \\ & - (f_i + 0.5\Delta_1 f_i)(s_i + 0.5\Delta_1 s_i) \\ & - 2(q_i + 0.5\Delta_1 q_i)(g_i + 0.5\Delta_1 g_i)] \end{aligned} \quad (29l)$$

$$\Delta_3 f_i = h(p_i + 0.5\Delta_2 p_i) \quad (29m)$$

$$\Delta_3 p_i = h(q_i + 0.5\Delta_2 q_i) \quad (29n)$$

$$\begin{aligned} \Delta_3 q_i = h\{ & \zeta[(p_i + 0.5\Delta_2 p_i)(r_i + 0.5\Delta_2 r_i) \\ & - (q_i + 0.5\Delta_2 q_i)(g_i + 0.5\Delta_2 g_i)] \\ & - (f_i + 0.5\Delta_2 f_i)(p_i + 0.5\Delta_2 p_i)\} \end{aligned} \quad (29o)$$

$$\Delta_3 g_i = h(r_i + 0.5\Delta_2 r_i) \quad (29p)$$

$$\Delta_3 r_i = h(s_i + 0.5\Delta_2 s_i) \quad (29q)$$

$$\begin{aligned} \Delta_3 s_i = h[& (p_i + 0.5\Delta_2 p_i)(r_i + 0.5\Delta_2 r_i) \\ & - (f_i + 0.5\Delta_2 f_i)(s_i + 0.5\Delta_2 s_i) \\ & - 2(q_i + 0.5\Delta_2 q_i)(g_i + 0.5\Delta_2 g_i)] \quad (29r) \end{aligned}$$

$$\Delta_4 f_i = h(p_i + \Delta_3 p_i) \quad (29s)$$

$$\Delta_4 p_i = h(q_i + \Delta_3 q_i) \quad (29t)$$

$$\begin{aligned} \Delta_4 q_i = h\{ & \zeta[(p_i + \Delta_3 p_i)(r_i + \Delta_3 r_i) \\ & - (q_i + \Delta_3 q_i)(g_i + \Delta_3 g_i)] \\ & - (f_i + \Delta_3 f_i)(p_i + \Delta_3 p_i)\} \quad (29u) \end{aligned}$$

$$\Delta_4 g_i = h(r_i + \Delta_3 r_i) \quad (29v)$$

$$\Delta_4 r_i = h(s_i + \Delta_4 s_i) \quad (29w)$$

$$\begin{aligned} \Delta_4 s_i = h[& (p_i + \Delta_3 p_i)(r_i + \Delta_3 r_i) \\ & - (f_i + \Delta_3 f_i)(s_i + \Delta_3 s_i) \\ & - 2(q_i + \Delta_3 q_i)(g_i + \Delta_3 g_i)]. \quad (29x) \end{aligned}$$

Using the values computed in equations (29 a-x), values for f , f' , f'' , g , g' and g'' are computed at $i+1$ by the following relations:

$$f_{i+1} = f_i + (\Delta_1 f_i + 2(\Delta_2 f_i + \Delta_3 f_i) + \Delta_4 f_i)/6.0 \quad (30a)$$

$$p_{i+1} = p_i + (\Delta_1 p_i + 2(\Delta_2 p_i + \Delta_3 p_i) + \Delta_4 p_i)/6.0 \quad (30b)$$

$$q_{i+1} = q_i + (\Delta_1 q_i + 2(\Delta_2 q_i + \Delta_3 q_i) + \Delta_4 q_i)/6.0 \quad (30c)$$

$$g_{i+1} = g_i + (\Delta_1 g_i + 2(\Delta_2 g_i + \Delta_3 g_i) + \Delta_4 g_i)/6.0 \quad (30d)$$

$$r_{i+1} = r_i + (\Delta_1 r_i + 2(\Delta_2 r_i + \Delta_3 r_i) + \Delta_4 r_i)/6.0 \quad (30e)$$

$$s_{i+1} = s_i + (\Delta_1 s_i + 2(\Delta_2 s_i + \Delta_3 s_i) + \Delta_4 s_i)/6.0. \quad (30f)$$

Similarly, to solve equations (26) and (27), a set of difference equations are derived as follows:

Let $\theta' = t$ and $\Phi' = u$

$$\Delta_1 \theta_i = ht_i \quad (31a)$$

$$\Delta_1 t_i = hPr[\zeta(p_i \Phi_i - t_i g_i) - f_i t_i] \quad (31b)$$

$$\Delta_1 \Phi_i = hu_i \quad (31c)$$

$$\Delta_1 u_i = hPr(p_i \Phi_i - f_i u_i - 2g_i t_i) \quad (31d)$$

$$\Delta_2 \theta_i = h(t_i + 0.5\Delta_1 t_i) \quad (31e)$$

$$\Delta_2 t_i = hPr\{\zeta[p_i(\Phi_i + 0.5\Delta_1 \Phi_i) - g_i(t_i + 0.5\Delta_1 t_i)] - f_i(t_i + 0.5\Delta_1 t_i)\} \quad (31f)$$

$$\Delta_2 \Phi_i = h(u_i + 0.5\Delta_1 u_i) \quad (31g)$$

$$\Delta_2 u_i = hPr[p_i(\Phi_i + 0.5\Delta_1 \Phi_i) - f_i(u_i + 0.5\Delta_1 u_i) - 2g_i(t_i + 0.5\Delta_1 t_i)] \quad (31h)$$

$$\Delta_3 \theta_i = h(t_i + 0.5\Delta_2 t_i) \quad (31i)$$

$$\Delta_3 t_i = hPr\{\zeta[p_i(\Phi_i + 0.5\Delta_2 \Phi_i) - g_i(t_i + 0.5\Delta_2 t_i)] - f_i(t_i + 0.5\Delta_2 t_i)\} \quad (31j)$$

$$\Delta_3 \Phi_i = h(u_i + 0.5\Delta_2 u_i) \quad (31k)$$

$$\Delta_3 u_i = hPr[p_i(\Phi_i + 0.5\Delta_2 \Phi_i) - f_i(u_i + 0.5\Delta_2 u_i) - 2g_i(t_i + 0.5\Delta_2 t_i)] \quad (31l)$$

$$\Delta_4 \theta_i = h(t_i + \Delta_3 t_i) \quad (31m)$$

$$\Delta_4 t_i = hPr\{\zeta[p_i(\Phi_i + \Delta_3 \Phi_i) - g_i(t_i + \Delta_3 t_i)] - f_i(t_i + \Delta_3 t_i)\} \quad (31n)$$

$$\Delta_4 \Phi_i = h(u_i + \Delta_3 u_i) \quad (31o)$$

$$\Delta_4 u_i = hPr[p_i(\Phi_i + \Delta_3 \Phi_i) - f_i(u_i + \Delta_3 u_i) - 2g_i(t_i + \Delta_3 t_i)] \quad (31p)$$

Using equations (31 a-p), θ , θ' , ϕ , and ϕ' at $i+1$ are obtained by:

$$\theta_{i+1} = \theta_i + (\Delta_1\theta_i + 2(\Delta_2\theta_i + \Delta_3\theta_i) + \Delta_4\theta_i)/6.0 \quad (32a)$$

$$t_{i+1} = t_i + (\Delta_1t_i + 2(\Delta_2t_i + \Delta_3t_i) + \Delta_4t_i)/6.0 \quad (32b)$$

$$\phi_{i+1} = \phi_i + (\Delta_1\phi_i + 2(\Delta_2\phi_i + \Delta_3\phi_i) + \Delta_4\phi_i)/6.0 \quad (32c)$$

$$u_{i+1} = u_i + (\Delta_1u_i + 2(\Delta_2u_i + \Delta_3u_i) + \Delta_4u_i)/6.0. \quad (32d)$$

Equations (29 a-x), (30 a-f), (31 a-p) and (32 a-d) are then coded in FORTRAN and input into an IBM XT. Equations (29 a-x) and (30 a-f) are solved first by assuming initial values for $f''(0)$ and $g''(0)$. After computing values for $f'(etamax)$ and $g'(etamax)$, they are compared to the known values of $f'(\infty)$ and $g'(\infty)$, respectively. If those values are within ϵ of each other, the iteration is stopped and the program continues on to solve equations (31 a-p) and (32 a-d). If $f'(etamax)$ or $g'(etamax)$ are not within ϵ , then new values of $f''(0)$ or $g''(0)$ are assumed. Again, to make more efficient use of the computer, the secant method is employed to make a new guess for the initial value. If after a large number of iterations (150), convergence did not occur, the case where the solution came closest to converging is used, and computation moves forward to solve equations (31 a-p) and (32 a-d). These equations are solved by the same method except $\theta'(0)$ and $\phi'(0)$ are assumed. For reasons which I am not able to ascertain, the solution to this set of equations is not achievable. The velocity

profiles for some representative cases are shown in Figures 17 - 20 with the coefficient of friction shown in Figure 21.

As is seen from Figures 17 - 20, the boundary layer thickens with increasing ζ and velocity difference $|U_\infty - U_w|$, as was seen in the similarity case.

Comparing values from Figure 21 at $U_w - U_\infty = -1.0$ with Figure 3 of Sparrow, Quack and Boerner [7] shows good agreement. From Figure 21 it is seen that separation occurs for $U_w - U_\infty = -1.0$ when ζ is approximately 0.4. It is also observed that the friction coefficient is greater when $U_w > U_\infty$ than when $U_w < U_\infty$. Also in Figure 21 it is seen that at $U_w - U_\infty = -0.6$ the curves for $\zeta = -0.2$ and -0.1 meet. This is due to round-off error.

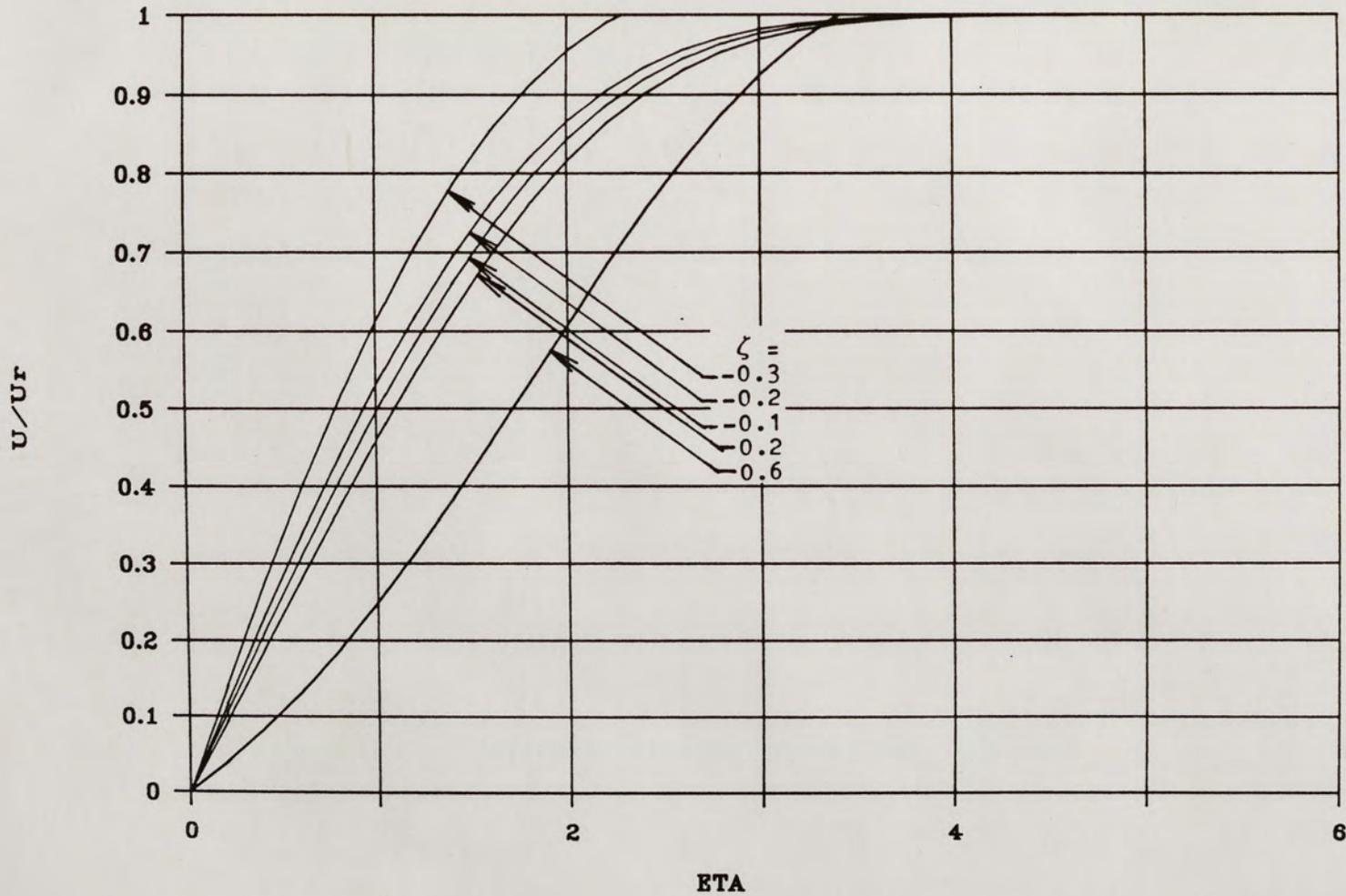


Figure 17. Velocity Profiles for $U_\infty - U_w = 1$ as a Function of ζ .

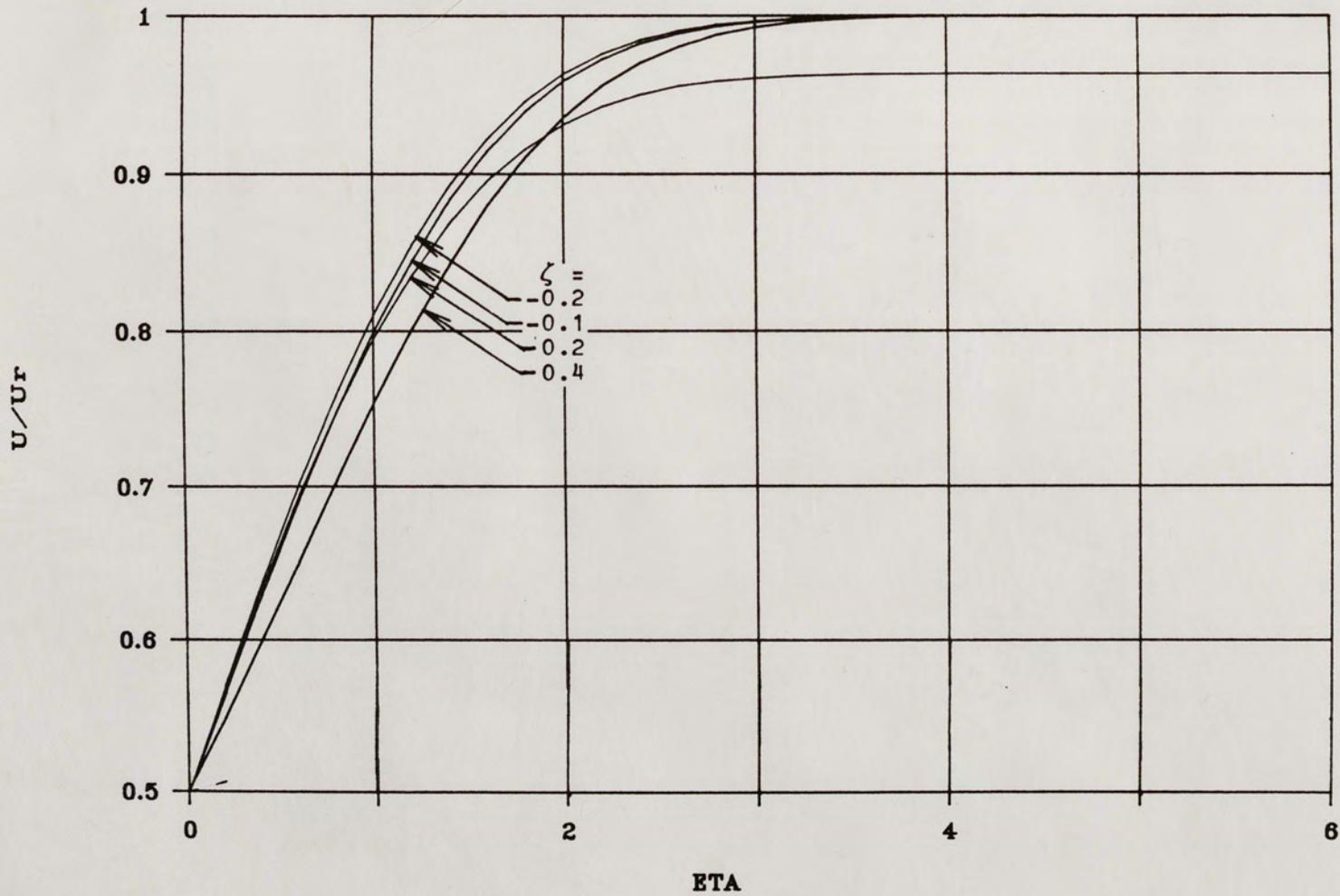


Figure 18. Velocity Profiles for $U_\infty - U_w = 0.5$.

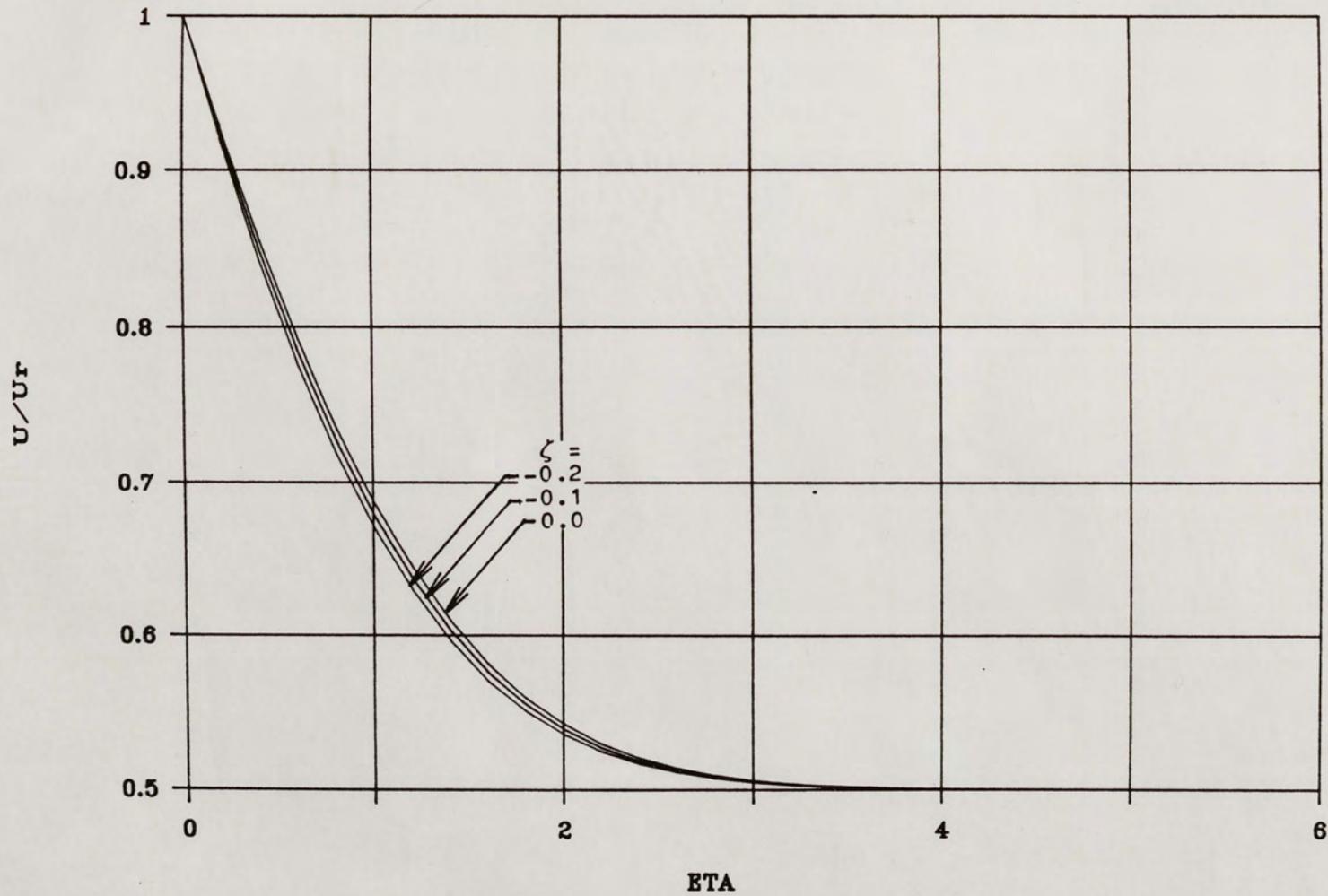


Figure 19. Velocity Profiles for $U_w - U_\infty = 0.5$.

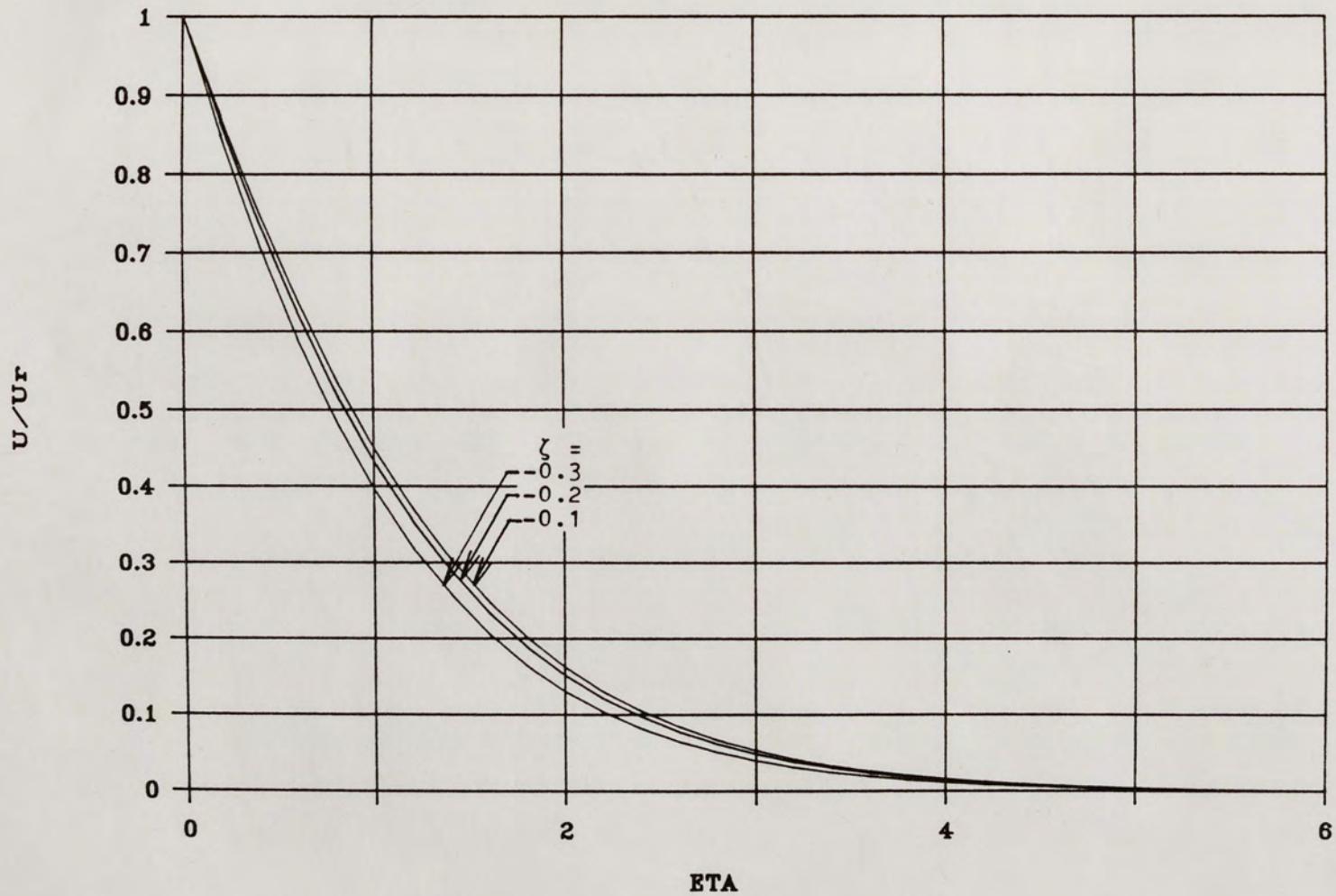


Figure 20. Velocity Profiles for $U_w - U_\infty = 1$.

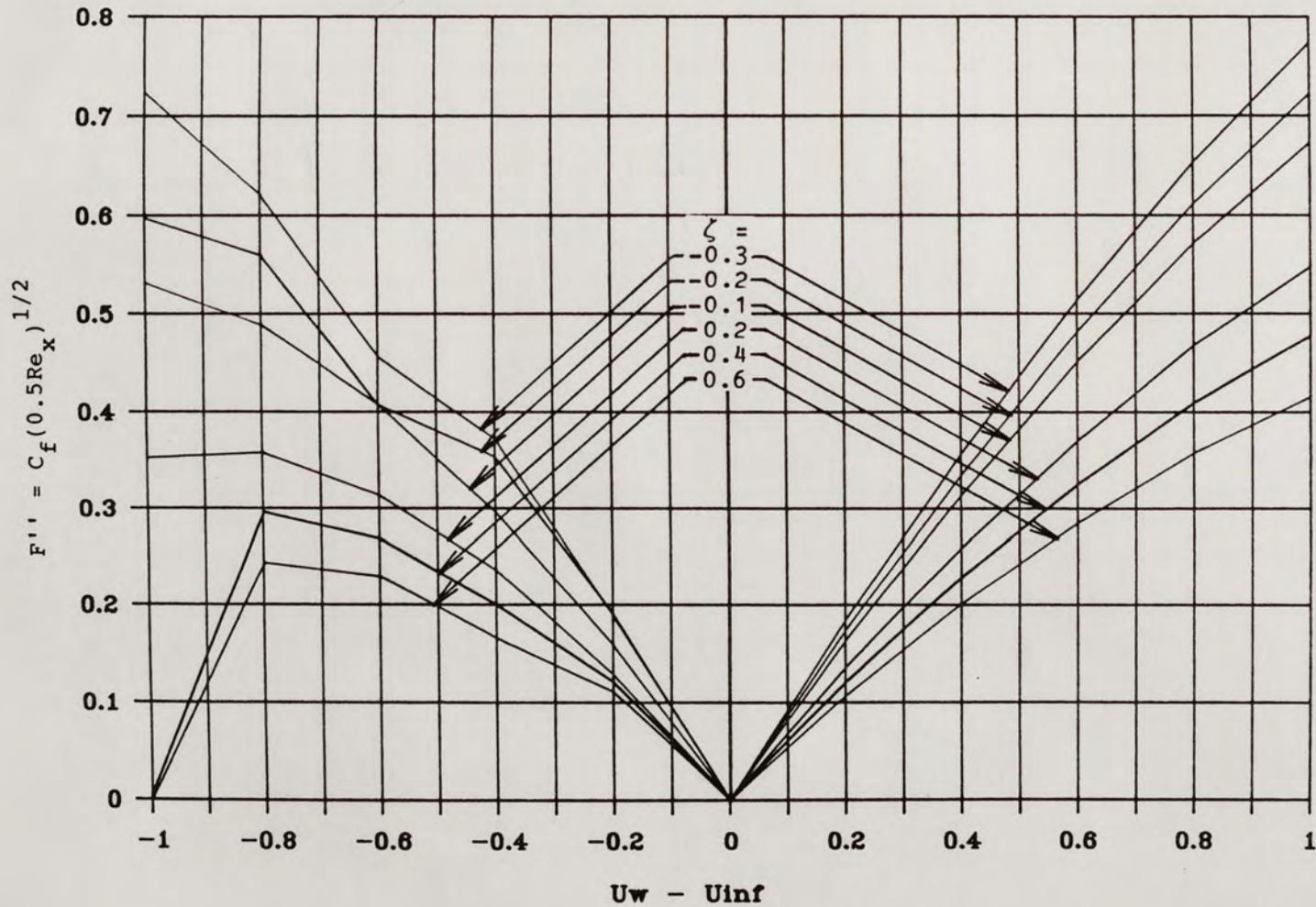


Figure 21. Friction Coefficient vs. ζ and Velocity Difference.

SUMMARY

From the analysis that has been performed, the velocity profiles show similar trends. In both the similar and non-similar boundary layer cases, as the velocity difference $|U_w - U_\infty|$ and relevant injection parameter, C or ζ , increase, the boundary layer thickness also increases. It was also observed that in both cases the shear stresses at the wall are greater when the wall velocity is greater than the free stream velocity, rather than when the free stream velocity is greater than the wall velocity while holding the velocity difference and injection parameter constant. Comparing the friction coefficient for the similar and non-similarity cases, it is seen that for the same values of the injection parameter and velocity difference $U_w - U_\infty$ the non-similarity is greater for BL suction and less for BL injection. It is also seen that separation for the non-similarity case occurs at an injection parameter, ζ , of about 0.4 which is approximately 35% less than for the corresponding similar case, where the injection parameter at separation is about 0.62.

For the solution of the energy equation, the similarity solutions showed that the Nusselt number was greatest for the case in which there was no velocity difference between

the plate and the free stream velocity. Also, the Nusselt number increased with increasing injection parameter, and decreased with increasing velocity difference. As was the case for the skin friction coefficient, the Nusselt number was greater for cases where $U_w > U_\infty$ than when $U_\infty > U_w$. Due to problems encountered in the solution of the energy equation in the non-similarity cases, no useful data were obtained.

Suggested areas for future investigation are 1) solve the energy equation for the non-similarity case and 2) mass transfer from the top of the boundary layer instead of through the bottom of the boundary layer. The latter suggestion has application for problems in two-phase flows.

APPENDICES

APPENDIX A

PROGRAM LISTING FOR SIMILAR BOUNDARY
LAYER SOLUTION

```

D Line# 1      7
1 #DEEUG
2 PROGRAM VAUGHN
3
4 C ***** THIS PROGRAM USES THE SECANT METHOD AND RUNGE-KUTTA
5 C METHOD TO CALCULATE THE SOLUTION OF BLASIUS EQUATION
6 C FOR A MOVING FLAT PLATE WITH BOUNDARY LAYER INJECTION *****
7
8 REAL M, M1, NU
9 INTEGER ULIMIT
10 EXTERNAL FN
11 DIMENSION CO(200,4), THETA(201), TABLE(54,161)
12 COMMON /VECTOR/ ETA(201), F(201), P(201), Q(201)
13 COMMON /BOUND/ CI
14 OPEN(2,FILE='PRN')
15 OPEN(10,FILE='C:VAUGHN.PRN',STATUS = NEW )
16 WRITE(*,*) ENTER VALUE OF UPPER LIMIT
17 READ(*,*) ULIMIT
18 EPS = 1.0E-06
19 ✓ WRITE(*,*) ENTER VALUE FOR PRANDTL NO.
20 ✓ READ(*,*) PR
21 H = 0.05
22 INCR = IFIX(0.2/H) 4 = IFIX
23 N = IFIX(10./H) 200
24 NF1 = N + 1
25 DO 2 I = 4, 54
1 26 TABLE(I,1) = FLOAT(I - 4)/5
1 27 2 CONTINUE
28 DO 60 IDIF = ULIMIT, ULIMIT - 1, -1
1 29 DIFRNC = IDIF/10.
1 30 IFLAG = 0
1 31 UINF = 1.0
1 32 UWAL = 1.0 - DIFRNC
1 33 10 CONTINUE
1 34 DO 50 IC = 1, 10
1 35 GO TO (101, 102, 103, 104, 105, 106, 107, 108, 109, 110), IC
1 36 101 CI = -1.000
1 37 GO TO 15
1 38 102 CI = -0.750
1 39 GO TO 15
1 40 103 CI = -0.500
1 41 GO TO 15
1 42 104 CI = -0.250
1 43 GO TO 15
2 44 105 CI = 0.000
2 45 GO TO 15
2 46 106 CI = 0.125
2 47 GO TO 15
2 48 107 CI = 0.250
2 49 GO TO 15
2 50 108 CI = 0.375
2 51 GO TO 15
2 52 109 CI = 0.500
2 53 GO TO 15
2 54 110 CI = 0.619
2 55 15 CONTINUE
2 56 WRITE(10,2000) UINF, UWAL, CI
2 57 WRITE(*,2000) UINF, UWAL, CI
2 58 C = UINF
2 59 X1 = 0.1

```

```

D Line# 1      7
2      60      IF (UINF .LT. UWAL) X1 = -X1
2      61      CALL RUNGE(10.0, H, UWAL, X1, Y1, FN)
2      62      ITER = 1
2      63      X2 = 0.2
2      64      IF (UINF .LT. UWAL) X2 = -X2
2      65      20 CALL RUNGE(10.0, H, UWAL, X2, Y2, FN)
2      66      ITER = ITER + 1
2      67      MI = (Y2 - Y1) / (X2 - X1)
2      68      NU = C - Y2
2      69      IF (ABS(NU) .LT. EPS) GO TO 30
2      70      X1 = X2
2      71      X2 = X2 + NU / MI
2      72      IF ((X2 .LT. 0.0) .AND. (UINF .GT. UWAL)) X2 = 5.0E-05
2      73      IF ((X2 .GT. 0.0) .AND. (UINF .LT. UWAL)) X2 = -5.0E-05
2      74      Y1 = Y2
2      75      GO TO 20
2      76
2      77 C      ..... SOLVE THE ENERGY EQUATION (3). USE EQUATION (2.2.11)
2      78 C      FROM CHOW .....
2      79
2      80      30 CONTINUE
2      81
2      82      DO 40 I = 1, N
3      83      CO(1,1) = 1.0 - H*PR*F(I + 1)/4.
3      84      CO(1,2) = -2.0
3      85      CO(I,3) = 1.0 + H*PR*F(I + 1)/4.
3      86      40 CO(I,4) = -2.0*H*H*PR*Q(I)*Q(I)
2      87
2      88 C      ..... MODIFY COEFFICIENTS ACCORDING TO (3.4.11 - 14) OF CHOW TO
2      89 C      INCORPORATE BOUNDARY CONDITIONS .....
2      90
2      91      THETA(NF1) = 0.0
2      92      CO(1,4) = CO(1,4) - CO(1,1)
2      93      CO(1,1) = 0.0
2      94      CO(N,4) = CO(N,4)
2      95      CO(N,3) = 0.0
2      96
2      97 C      ..... COMPUTE THETA(I) FOR I = 1,2,...., N. ....
2      98
2      99      CALL TRID(CO, THETA, N)
2     100
2     101      IUL = NF1 - 2
2     102      DO 45 I = 0, IUL
3     103      J = NF1 - I
3     104      THETA(J) = THETA(J - 1)
3     105      45 CONTINUE
2     106      THETA(1) = 1.0
2     107
2     108 C      ..... PRINT THE NUMBER OF ITERATIONS AND THEN PRINT F, F*, AND F**
2     109 C      AS FUNCTIONS OF ETA. WHERE AN * INDICATES DIFFERENTIATION
2     110 C      WITH RESPECT TO ETA. THUS F* IS P AND F** IS Q .....
2     111
2     112      THETAP = (THETA(1) - THETA(2))/H
2     113      WRITE(*,230) THETAP
2     114      WRITE(10,230) THETAP
2     115      WRITE(*,200) ITER
2     116      WRITE(10,200) ITER
2     117      WRITE(*,210) (ETA(I),F(I),P(I),Q(I),THETA(I), I = 1,N+4,INCR)
2     118      WRITE(10,210) (ETA(I),F(I),P(I),Q(I),THETA(I), I = 1,N+4,INCR)

```

```

D Line# 1      7
2  119      WRITE(*,220)
2  120      WRITE(10,220)
2  121      50 CONTINUE
1  122      IF (IFLAG .EQ. 1) THEN
1  123          IFLAG = 0
1  124          GO TO 60
1  125          ELSE
1  126          IFLAG = 1
1  127          UWAL = 1.0
1  128          UINF = 1.0 - DIFRNC
1  129          GO TO 10
1  130          ENDIF
1  131      60 CONTINUE
132      200 FORMAT( 12X, I7 // 15X,3HETA, 10X, 1HF, 13X, 2HF*, 11X, 3HF**,
133          A 10X, 5HTHETA / 14X, 5H-----, 4(5X, 9H-----) )
134      210 FORMAT( 12X, F7.2, 4F14.7 )
135      220 FORMAT(1X)
136      230 FORMAT(12X, F10.7)
137      2000 FORMAT(1H1,11X, F7.2, F14.2 / 11X, F8.3)
138          CLOSE(UNIT=10)
139          STOP
140          END

```

Name	Type	Offset	P	Class
ABS				INTRINSIC
C	REAL	38856		
CI	REAL	0		/BOUND /
CO	REAL	35596		
DIFRNC	REAL	38836		
EPS	REAL	38800		
ETA	REAL	0		/VECTOR/
F	REAL	804		/VECTOR/
FLOAT				INTRINSIC
FN				EXTERNAL
H	REAL	38808		
I	INTEGER*4	38824		
IC	INTEGER*4	38852		
IDIF	INTEGER*4	38828		
IFIX				INTRINSIC
IFLAG	INTEGER*4	38840		
INCR	INTEGER*4	38812		
ITER	INTEGER*4	38868		
IUL	INTEGER*4	38892		
J	INTEGER*4	38900		
M	REAL	*****		
M1	REAL	38880		
N	INTEGER*4	38816		
NP1	INTEGER*4	38820		
NU	REAL	38884		
P	REAL	1608		/VECTOR/
PR	REAL	38804		
Q	REAL	2412		/VECTOR/
TABLE	REAL	820		
THETA	REAL	16		
THETA*	REAL	38904		
UINF	REAL	38844		
ULIMIT	INTEGER*4	38796		
UWAL	REAL	38848		

```

D Line# 1      7
x1      REAL      38860
x2      REAL      38872
y1      REAL      38864
y2      REAL      38876

```

```

141
142      FUNCTION FN(X, Y)
143      FN = -X * Y / 2.
144      RETURN
145      END

```

Name	Type	Offset	P	Class
x	REAL	0	*	
y	REAL	4	*	

```

146
147      SUBROUTINE RUNGE(ETAMAX, H, P0, Q0, Y, FNCTN)
148      COMMON /VECTOR/ ETA(201), F(201), Q(201)
149      COMMON /BOUND/ CI
150
151      I = 1
152      ETA(I) = 0.0
153      F(I) = -2.*CI
154      P(I) = P0
155      Q(I) = Q0
156
157 C      ..... APPLY RUNGE-KUTTA FORMULAE (9a - 9f) AND
158 C      (10a - 10c) OF CHOW UNTIL ETAMAX IS REACHED .....
159      10 D1F = H*F(I)
160      D1Q = H*Q(I)
161      D10 = H*FNCTN(F(I), Q(I))
162
163      D2F = H*(F(I)+D1F/2.)
164      D2Q = H*(Q(I)+D1Q/2.)
165      D20 = H*FNCTN(F(I) + D1F/2., Q(I) + D1Q/2.)
166
167      D3F = H*(F(I)+D2F/2.)
168      D3Q = H*(Q(I)+D2Q/2.)
169      D30 = H*FNCTN(F(I) + D2F/2., Q(I) + D2Q/2.)
170
171      D4F = H*(F(I)+D3F)
172      D4Q = H*(Q(I)+D3Q)
173      D40 = H*FNCTN(F(I) + D3F, Q(I) + D3Q)
174
175      F(I+1) = F(I) + (D1F + 2.*(D2F + D3F) + D4F)/6.0
176      Q(I+1) = Q(I) + (D1Q + 2.*(D2Q + D3Q) + D4Q)/6.0
177      Q(I+1) = Q(I) + (D10 + 2.*(D20 + D30) + D40)/6.0
178      ETA(I+1) = ETA(I) + H
179      I = I + 1
180      IF( ETA(I)-ETAMAX ) 10, 20, 20
181
182 C      ..... KEEP ITERATING UNTIL THE APPROXIMATE BOUNDARY CONDITION
183 C      (3.3.18) IS SATISFIED. HALF-INTERVAL METHOD IS USED TO
184 C      MODIFY THE STARTING VALUE OF Q .....
185
186      20 Y = F(I)

```

```
D Line# 1      7
      187      RETURN
      188      END
```

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Name	Type	Offset	P	Class
CI	REAL	0		/BOUND /
D1F	REAL	39106		
D1P	REAL	39110		
D1Q	REAL	39114		
D2F	REAL	39122		
D2P	REAL	39126		
D2Q	REAL	39130		
D3F	REAL	39138		
D3P	REAL	39142		
D3Q	REAL	39146		
D4F	REAL	39154		
D4P	REAL	39158		
D4Q	REAL	39162		
ETA	REAL	0		/VECTOR/
ETAMAX	REAL	0	*	
F	REAL	804		/VECTOR/
FNCTN				SUB/FUN PARM
H	REAL	4	*	
I	INTEGER*4	39102		
P	REAL	1608		/VECTOR/
PQ	REAL	8	*	
Q	REAL	2412		/VECTOR/
QQ	REAL	12	*	
Y	REAL	16	*	

```

189
190      SUBROUTINE TRID(C, F, N)
191 C      ***** SOLVING A SET OF LINEAR SIMULTANOUS EQUATIONS DESCRIBED
192 C          BY (2.2.13) HAVING A TRIDIAGONAL COEFFICIENT MATRIX.
193 C          COEFFICIENT MATRICES ARE REPRESENTED BY C(I,J) AND THE
194 C          SOLUTION IS STORED IN THE ONE-DIMENSIONAL ARRAY F(I) *****
195
196      DIMENSION C(200,4), F(201)
197
198 C      ***** ELIMINATE THE FIRST TERM OF EACH OF THE EQUATIONS (EXCEPT
199 C          THE FIRST AND LAST ONE) ACCORDING TO (2.2.14) *****
200      NM1 = N - 1
201      DO 1 I = 2, NM1
1 202          C(I,2) = C(I,2) * C(I-1,2) - C(I,1) * C(I-1,3)
1 203          C(I,3) = C(I,3) * C(I-1,2)
1 204          C(I,4) = C(I,4) * C(I-1,2) - C(I,1) * C(I-1,4)
205
206 C      ***** COMPUTE SOLUTION USING (2.2.15) AND (2.2.16) *****
207
208          F(N) = ( C(N,4) * C(N-1,2) - C(N,1) * C(N-1,4) ) /
209          A      ( C(N,2) * C(N-1,2) - C(N,1) * C(N-1,3) )
210          DO 2 K = 1, NM1
1 211              J = N - K
1 212              F(J) = ( C(J,4) - C(J,3) * F(J+1) ) / C(J,2)
213          RETURN
214          END
```

D Line# 1 7

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Name	Type	Offset	P	Class
C	REAL	0	*	
F	REAL	4	*	
I	INTEGER*4	39178		
J	INTEGER*4	39194		
K	INTEGER*4	39186		
N	INTEGER*4	8	*	
NM1	INTEGER*4	39174		

Name	Type	Size	Class
BOUND		4	COMMON
FN	REAL		FUNCTION
RUNGE			SUBROUTINE
TR1D			SUBROUTINE
VAUGHN			PROGRAM
VECTOR		3216	COMMON

Pass One No Errors Detected
214 Source Lines

APPENDIX B

SAMPLE OUTPUT FOR SIMILAR BOUNDARY
LAYER PROBLEM

FREESTREAM VELOCITY = 1.00 VELOCITY AT THE WALL = .00
 INJECTION PARAMETER (C) = 0.000: PR = .72
 NUMBER OF ITERATIONS = 6

ETA	F	F*	F**	THETA
----	-----	-----	-----	-----
0.00	0.0000000	.0000001	.3320573	.8477594
.20	.0066410	.0664078	.3319838	.8445846
.40	.0265599	.1327642	.3314698	.8350683
.60	.0597347	.1989373	.3300791	.8192528
.80	.1061083	.2647092	.3273892	.7972509
1.00	.1655718	.3297800	.3230071	.7692761
1.20	.2379487	.3937761	.3165892	.7356709
1.40	.3229816	.4562617	.3078654	.6969256
1.60	.4203208	.5167567	.2966634	.6536885
1.80	.5295181	.5747581	.2829310	.6067604
2.00	.6500244	.6297657	.2667515	.5570766
2.20	.7811934	.6813104	.2483509	.5056716
2.40	.9222901	.7289819	.2280917	.4536329
2.60	1.0725060	.7724550	.2064546	.4020444
2.80	1.2309774	.8115095	.1840066	.3519304
3.00	1.3968083	.8460444	.1613603	.3042013
3.20	1.5690949	.8760813	.1391280	.2596107
3.40	1.7469500	.9017611	.1178762	.2187260
3.60	1.9295250	.9233295	.0980863	.1819172
3.80	2.1160297	.9411178	.0801259	.1493605
4.00	2.3057461	.9555181	.0642341	.1210567
4.20	2.4980392	.9669569	.0505197	.0968588
4.40	2.6923604	.9758707	.0389726	.0765055
4.60	2.8882475	.9826833	.0294838	.0596558
4.80	3.0853202	.9877893	.0218712	.0459216
5.00	3.2832732	.9915417	.0159068	.0348962
5.20	3.4818671	.9942454	.0113418	.0261770
5.40	3.6809185	.9961551	.0079277	.0193833
5.60	3.8802900	.9974776	.0054320	.0141670
5.80	4.0798812	.9983753	.0036484	.0102198
6.00	4.2796197	.9989727	.0024020	.0072761
6.20	4.4794559	.9993624	.0015502	.0051123
6.40	4.6793551	.9996115	.0009806	.0035446
6.60	4.8792944	.9997677	.0006080	.0024250
6.80	5.0792584	.9998636	.0003696	.0016370
7.00	5.2792373	.9999213	.0002202	.0010902
7.20	5.4792256	.9999554	.0001286	.0007163
7.40	5.6792188	.9999751	.0000736	.0004642
7.60	5.8792152	.9999864	.0000413	.0002967
7.80	6.0792131	.9999926	.0000227	.0001870
8.00	6.2792120	.9999960	.0000122	.0001162
8.20	6.4792109	.9999979	.0000065	.0000712
8.40	6.6792097	.9999988	.0000034	.0000429
8.60	6.8792095	.9999993	.0000017	.0000254
8.80	7.0792103	.9999996	.0000008	.0000148
9.00	7.2792110	.9999997	.0000004	.0000084
9.20	7.4792118	.9999997	.0000002	.0000046
9.40	7.6792126	.9999997	.0000001	.0000024
9.60	7.8792133	.9999997	.0000000	.0000011
9.80	8.0792141	.9999997	.0000000	.0000004
10.00	8.2792149	.9999997	.0000000	0.0000000

APPENDIX C

PROGRAM LISTING FOR NON-SIMILAR BOUNDARY
LAYER SOLUTION

```

D Line# 1      7
1 #DEBUG
2 PROGRAM CONTHS
3
4 C ***** THIS PROGRAM USES THE SECANT METHOD AND RUNGE-KUTTA
5 C METHOD TO CALCULATE THE SOLUTION OF BLASIOUS' EQUATION
6 C FOR A MOVING FLAT PLATE WITH BOUNDARY LAYER INJECTION *****
7
8 IMPLICIT REAL*8 (A-G,O-Z)
9 REAL*8 MIF, MIG, NUF, NUG
10 INTEGER ULIMIT
11 EXTERNAL FF, FG
12 DIMENSION BEST(4)
13 COMMON /VECTOR/ ETA(201), F(201), P(201), Q(201),
14 1 G(201), R(201), S(201)
15 COMMON /BOUND/ ZETA
16 COMMON /FRANTL/ PR
17 COMMON ITRUBL
18 DATA ITMAX1, ITMAX2/15, 150/
19 OPEN(2,FILE='PRN')
20 OPEN(10,FILE = 'CONTHS.PRN', STATUS = 'NEW')
21 DO 5 I = 1, 201
1 22 ETA(I) = 0.0
1 23 F(I) = 0.0
1 24 P(I) = 0.0
1 25 Q(I) = 0.0
1 26 G(I) = 0.0
1 27 R(I) = 0.0
1 28 S(I) = 0.0
1 29 5 CONTINUE
30 WRITE(*,*) ' ENTER VALUE OF UPPER LIMIT '
31 READ(*,*) ULIMIT
32 ETAMAX = 6.0
33 EPS = 5.0E-06
34 WRITE(*,*) ' ENTER A VALUE FOR THE PRANDTL NUMBER.'
35 READ(*,*) PR
36 DO 110 IDIF = ULIMIT, ULIMIT - 2, -2
1 37 DIFRNC = IDIF/10.
1 38 IFLAG = 0
1 39 UINF = 1.0
1 40 UWAL = 1.0 - DIFRNC
1 41 10 CONTINUE
1 42 DO 100 IC = 1, 7
2 43 XF1 = 0.6
2 44 XG1 = -0.7
2 45 XT1 = -0.5
2 46 XP1 = 0.1
2 47 GO TO (114, 115, 116, 117, 118, 119, 120), IC
2 48 114 ZETA = -0.3
2 49 H = 0.05
2 50 INCR = IFIX(0.2/H)
2 51 N = IFIX(6.0/H)
2 52 NP1 = N + 1
2 53 GO TO 15
2 54 115 ZETA = -0.2
2 55 H = 0.05
2 56 INCR = IFIX(0.2/H)
2 57 N = IFIX(6.0/H)
2 58 NP1 = N + 1
2 59 GO TO 15

```

```

D Line# 1      7
2      60    116 ZETA = -0.1
2      61      H = 0.1
2      62      INCR = IFIX(0.2/H)
2      63      N = IFIX(6.0/H)
2      64      NP1 = N + 1
2      65      GO TO 15
2      66    117 ZETA = 0.0
2      67      H = 0.1
2      68      INCR = IFIX(0.2/H)
2      69      N = IFIX(6.0/H)
2      70      NP1 = N + 1
2      71      GO TO 15
2      72    118 ZETA = 0.2
2      73      H = 0.1
2      74      INCR = IFIX(0.2/H)
2      75      N = IFIX(6.0/H)
2      76      NP1 = N + 1
2      77      GO TO 15
2      78    119 ZETA = 0.4
2      79      H = 0.05
2      80      INCR = IFIX(0.2/H)
2      81      N = IFIX(6.0/H)
2      82      NP1 = N + 1
2      83      GO TO 15
2      84    120 ZETA = 0.6
2      85      H = 0.05
2      86      INCR = IFIX(0.2/H)
2      87      N = IFIX(6.0/H)
2      88      NP1 = N + 1
2      89      15 CONTINUE
2      90      WRITE(2,2000) UINF, UWAL, ZETA, PR
2      91      WRITE(*,2000) UINF, UWAL, ZETA, PR
2      92      CONST = UINF
2      93      IF (UINF .LT. UWAL) XF1 = -ABS(XF1)
2      94      CALL RUNGE(ETAMAX, H, UWAL, XF1, XG1, YF1, YG1, FF, FG)
2      95      ERRORV = ABS(CONST - YF2) + ABS(YG2)
2      96      BEST(1) = XF1
2      97      BEST(2) = XG1
2      98      BEST(3) = CONST - YF1
2      99      BEST(4) = -YG1
2     100      ITER = 1
2     101      ITERR = 1
2     102      XF2 = 0.9*XF1
2     103      XG2 = 0.9*XG1
2     104      20 CONTINUE
2     105      CALL RUNGE(ETAMAX, H, UWAL, XF2, XG2, YF2, YG2, FF, FG)
2     106      WRITE(*,300) XF2, YF2, XG2, YG2
2     107      ITER = ITER + 1
2     108      ITERR = ITERR + 1
2     109      IF (ITERR .GE. ITMAX1) THEN
2     110      ITER = 0
2     111      XF2 = XF2 + 0.01
2     112      XG2 = XG2 + 0.01
2     113      GO TO 30
2     114      ENDIF
2     115      IF (ITRUBL .EQ. 1) THEN
2     116      ITRUBL = 0
2     117      XF2 = 0.5*(XF2 + XF1)
2     118      XG2 = 0.5*(XG2 + XG1)

```

```

D Line# 1      7
2 119          ITERR = 0
2 120          WRITE(*,*) ' RETURN A
2 121          GO TO 40
2 122          ENDIF
2 123          MIF = (YF2 - YF1) / (XF2 - XF1)
2 124          NUF = CONST - YF2
2 125          NUG = -YG2
2 126          XF1 = XF2
2 127          YF1 = YF2
2 128          XF2 = XF2 + NUF / MIF
2 129          IF ((ABS(NUF) .LT. EPS) .AND. (ABS(NUG) .LT. EPS)) GO TO 50
2 130          IF (ERRORV .GE. ABS(NUF) + ABS(NUG)) THEN
2 131          ERRORV = ABS(NUF) + ABS(NUG)
2 132          BEST(1) = XF2
2 133          BEST(2) = XG2
2 134          BEST(3) = NUF
2 135          BEST(4) = NUG
2 136          ENDIF
2 137          IF (ABS(NUF) .LT. EPS) THEN
2 138          ITERR = 0
2 139          WRITE(*,*) ' RETURN B
2 140          GO TO 30
2 141          ENDIF
2 142          IF (ITMAX1 .EQ. ITERR) THEN
2 143          ITERR = 0
2 144          WRITE(*,*) ' RETURN C
2 145          GO TO 30
2 146          ENDIF
2 147          IF (ITER .GT. ITMAX2) THEN
2 148          GO TO 40
2 149          ELSE
2 150          GO TO 20
2 151          ENDIF
2 152          30 CONTINUE
2 153          CALL RUNGE(ETAMAX, H, UWAL, XF2, XG2, YF2, YG2, FF, FG)
2 154          WRITE(*,300) XF2, YF2, XG2, YG2
2 155          ITER = ITER + 1
2 156          ITERR = ITERR + 1
2 157          IF ((YG2 .EQ. YG1) .OR. (XG2 .EQ. XG1)) THEN
2 158          ITERR = 0
2 159          WRITE(*,*) ' RETURN 1
2 160          GO TO 40
2 161          ENDIF
2 162          MIG = (YG2 - YG1) / (XG2 - XG1)
2 163          NUF = CONST - YF2
2 164          NUG = -YG2
2 165          XG1 = XG2
2 166          YG1 = YG2
2 167          XG2 = XG2 + NUG / MIG
2 168          IF (XG2 .GT. XG1 + 2.0) XG2 = XG1 + 2.0
2 169          IF ((ABS(NUF) .LT. EPS) .AND. (ABS(NUG) .LT. EPS)) GO TO 50
2 170          IF (ERRORV .GE. ABS(NUF) + ABS(NUG)) THEN
2 171          ERRORV = ABS(NUF) + ABS(NUG)
2 172          BEST(1) = XF2
2 173          BEST(2) = XG2
2 174          BEST(3) = NUF
2 175          BEST(4) = NUG
2 176          ENDIF
2 177          IF (ABS(NUG) .LT. EPS) THEN

```

```

D Line# 1      7
2 178      BEST(4) = NUG
2 179      ENDIF
2 180      IF (ABS(NUG) .LT. EPS) THEN
2 181          ITERR = 0
2 182          WRITE(*,*)      RETURN 2
2 183          GO TO 20
2 184      ENDIF
2 185      IF (ITERR .EQ. ITMAX1) THEN
2 186          ITERR = 0
2 187          WRITE(*,*)      RETURN 3
2 188          GO TO 20
2 189      ENDIF
2 190      IF (ITER .GT. ITMAX2) THEN
2 191          ITERR = 0
2 192          GO TO 40
2 193      ELSE
2 194          GO TO 30
2 195      ENDIF
2 196      40 CONTINUE
2 197          WRITE(*,240) BEST(3), BEST(4)
2 198          WRITE(2,240) BEST(3), BEST(4)
2 199          CALL RUNGE(ETAMAX, H, UWAL, BEST(1), BEST(2), YF2, YG2, FF, FG)
2 200      50 CONTINUE
2 201
2 202          WRITE(*,200)
2 203          WRITE(2,200)
2 204          WRITE(*,210) (ETA(I),F(I),P(I),Q(I), I=1,N+INCR,INCR)
2 205          WRITE(2,210) (ETA(I),F(I),P(I),Q(I), I=1,N+INCR,INCR)
2 206          WRITE(10,210) (ETA(I),F(I),P(I),Q(I), I=1,N+INCR,INCR)
2 207          WRITE(*,220)
2 208          WRITE(2,220)
2 209          XF2 = 0.9*XF2
2 210          XG2 = 0.9*XG2
2 211          XT2 = 0.9*XT2
2 212          XP2 = 0.9*XP2
2 213
2 214      100 CONTINUE
1 215          IF (IFLAG .EQ. 1) THEN
1 216              IFLAG = 0
1 217              GO TO 110
1 218          ELSE
1 219              IFLAG = 1
1 220              UWAL = 1.0
1 221              UINF = 1.0 - DIFRNC
1 222              GO TO 10
1 223          ENDIF
1 224      110 CONTINUE
225
226          CLOSE(10)
227
228      200 FORMAT( 1H0, 15X,3HETA, 10X, 1HF, 13X, 2HF*, 11X, 3HF** / 14X,
229      A      5H-----, 3(5X, 9H-----) )
230      210 FORMAT( 12X, F7.2, 3F14.7 )
231      220 FORMAT(1X)
232      230 FORMAT(12X, F10.7)
233      240 FORMAT(1H0, 11X, 'CONVERGENCE DID NOT OCCUR.' / 12X,
234      A      'ERROR FOR F =', F10.6/ 12X, 'ERROR FOR G =', F10.6)
235      250 FORMAT(1H0, 11X, 'CONVERGENCE DID NOT OCCUR.' / 12X,
236      A      'ERROR FOR THETA =', F10.6/ 12X, 'ERROR FOR PHI =', F10.6)

```

```

D Line# 1      7
237      A          YP2 = , F10.5)
238 2000 FORMAT(1H1,10X, ' FREESTREAM VELOCITY = ', F7.2, ' VELOCITY AT THE
239      A WALL = ', F7.2 /, 12X, ' INJECTION PARAMETER (ZETA) = ', F8.3)
240      STOP
241      END

```

Name	Type	Offset	P	Class
ABS				INTRINSIC
BEST	REAL*8	20		
CONST	REAL*8	172		
DIFRNC	REAL*8	92		
EPS	REAL*8	76		
ERRORV	REAL*8	196		
ETA	REAL*8	0		/VECTOR/
ETAMAX	REAL*8	68		
F	REAL*8	1608		/VECTOR/
FF				EXTERNAL
FG				EXTERNAL
G	REAL*8	6432		/VECTOR/
H	REAL	156		
I	INTEGER*4	60		
IC	INTEGER*4	120		
IDIF	INTEGER*4	84		
IFIX				INTRINSIC
IFLAG	INTEGER*4	100		
INCR	INTEGER*4	160		
ITER	INTEGER*4	220		
ITERR	INTEGER*4	224		
ITMAX1	INTEGER*4	52		
ITMAX2	INTEGER*4	56		
ITRUBL	INTEGER*4	0		/COMMQU/
MIF	REAL*8	244		
MIG	REAL*8	268		
N	INTEGER*4	164		
NF1	INTEGER*4	168		
NUF	REAL*8	252		
NUG	REAL*8	260		
P	REAL*8	3216		/VECTOR/
Q	REAL*8	4824		/VECTOR/
R	REAL*8	8040		/VECTOR/
S	REAL*8	9648		/VECTOR/
UINF	REAL*8	104		
ULIMIT	INTEGER*4	64		
UWAL	REAL*8	112		
XF1	REAL*8	124		
XF2	REAL*8	228		
XG1	REAL*8	132		
XG2	REAL*8	236		
XF1	REAL*8	148		
XF2	REAL*8	292		
XT1	REAL*8	140		
XT2	REAL*8	284		
YF1	REAL*8	180		
YF2	REAL*8	204		
YG1	REAL*8	188		
YG2	REAL*8	212		
ZETA	REAL*8	0		/BOUND /

```

D Line# 1      7
 243      FUNCTION FF(V, W, X, Y, Z)
 244      IMPLICIT REAL*8 (A-G,O-Z)
 245      COMMON /BOUND/ ZETA
 246      COMMON ITRUBL
 247      FF = ZETA*(X*Z - Y*W) - V*Y
 248      IF (ABS(FF) .LT. 100.0) GO TO 10
 249      FF = 0.0
 250      ITRUBL = 1
 251      10 CONTINUE
 252      RETURN
 253      END

```

Name	Type	Offset	P	Class
ABS				INTRINSIC
ITRUBL	INTEGER*4	0		/COMMON/
V	REAL*8	0	*	
W	REAL*8	4	*	
X	REAL*8	8	*	
Y	REAL*8	12	*	
Z	REAL*8	16	*	
ZETA	REAL*8	0		/BOUND /

```

254
255      FUNCTION FG(U, V, W, X, Y, Z)
256      IMPLICIT REAL*8 (A-G,O-Z)
257      FG = W*Y - U*Z - 2.0*X*V
258      RETURN
259      END

```

Name	Type	Offset	P	Class
U	REAL*8	0	*	
V	REAL*8	4	*	
W	REAL*8	8	*	
X	REAL*8	12	*	
Y	REAL*8	16	*	
Z	REAL*8	20	*	

```

260
261      SUBROUTINE RUNGE(ETAMAX, H, P0, Q0, S0, YF, YG, FNC, GNC)
262      IMPLICIT REAL*8 (A-G,O-Z)
263      COMMON /VECTOR/ ETA(201), F(201), P(201), Q(201),
264      1      G(201), R(201), S(201)
265      COMMON /BOUND/ ZETA
266      COMMON ITRUBL
267
268      I = 1
269      ETA(I) = 0.0
270      F(I) = - ZETA/2.0
271      P(I) = P0
272      Q(I) = Q0
273      G(I) = -0.5
274      R(I) = 0.0
275      S(I) = S0
276
277 C      ..... APPLY RUNGE-KUTTA FORMULAE (29a - 29e) AND

```

```

D Line# 1      7
278 C          (30a through 30f) UNTIL ETAMAX IS REACHED .....
279          10 D1F = H*F(I)
280             D1P = H*Q(I)
281             D1Q = H*FNC(F(I), G(I), P(I), Q(I), R(I))
282             D1G = H*R(I)
283             D1R = H*S(I)
284             D1S = H*GNC(F(I), G(I), P(I), Q(I), R(I), S(I))
285
286             D2F = H*(F(I) + D1P/2.)
287             D2P = H*(Q(I) + D1Q/2.)
288             D2Q = H*FNC(F(I) + D1F/2., G(I) + D1G/2., F(I) + D1P/2.,
289 A             Q(I) + D1Q/2., R(I) + D1R/2.)
290             D2G = H*(R(I) + D1R/2.)
291             D2R = H*(S(I) + D1S/2.)
292             D2S = H*GNC(F(I) + D1F/2., G(I) + D1G/2., P(I) + D1P/2.,
293 A             Q(I) + D1Q/2., R(I) + D1R/2., S(I) + D1S/2.)
294
295             D3F = H*(P(I) + D2P/2.)
296             D3P = H*(Q(I) + D2Q/2.)
297             D3Q = H*FNC(F(I) + D2F/2., G(I) + D2G/2., F(I) + D2P/2.,
298 A             Q(I) + D2Q/2., R(I) + D2R/2.)
299             D3G = H*(R(I) + D2R/2.)
300             D3R = H*(S(I) + D2S/2.)
301             D3S = H*GNC(F(I) + D2F/2., G(I) + D2G/2., P(I) + D2P/2.,
302 A             Q(I) + D2Q/2., R(I) + D2R/2., S(I) + D2S/2.)
303
304             D4F = H*(P(I) + D3P)
305             D4P = H*(Q(I) + D3Q)
306             D4Q = H*FNC(F(I) + D3F, G(I) + D3G, F(I) + D3P,
307 A             Q(I) + D3Q, R(I) + D3R)
308             D4G = H*(R(I) + D3R)
309             D4R = H*(S(I) + D3S)
310             D4S = H*GNC(F(I) + D3F, G(I) + D3G, F(I) + D3P,
311 A             Q(I) + D3Q, R(I) + D3R, S(I) + D3S)
312
313             F(I+1) = F(I) + (D1F + 2.*(D2F + D3F) + D4F)/6.0
314             P(I+1) = P(I) + (D1P + 2.*(D2P + D3P) + D4P)/6.0
315             Q(I+1) = Q(I) + (D1Q + 2.*(D2Q + D3Q) + D4Q)/6.0
316             G(I+1) = G(I) + (D1G + 2.*(D2G + D3G) + D4G)/6.0
317             R(I+1) = R(I) + (D1R + 2.*(D2R + D3R) + D4R)/6.0
318             S(I+1) = S(I) + (D1S + 2.*(D2S + D3S) + D4S)/6.0
319
320             IF (ITRUBL .EQ. 1) GO TO 20
321             ETA(I+1) = ETA(I) + H
322             I = I + 1
323             IF( ETA(I)-ETAMAX ) 10, 20, 20
324
325 C          ..... KEEP ITERATING UNTIL THE APPROXIMATE BOUNDARY CONDITION
326 C          (3.3.18) IS SATISFIED. HALF-INTERVAL METHOD IS USED TO
327 C          MODIFY THE STARTING VALUE OF Q .....
328
329          20 YF = P(I)
330             YG = R(I)
331             RETURN
332             END

```

Name	Type	Offset	P	Class
------	------	--------	---	-------

D1F	REAL*8	980		
-----	--------	-----	--	--

```

D Line# 1      7
D1G  REAL*8    1012
D1P  REAL*8    988
D1Q  REAL*8    996
D1R  REAL*8   1020
D1S  REAL*8   1028
D2F  REAL*8   1044
D2G  REAL*8   1076
D2P  REAL*8   1052
D2Q  REAL*8   1060
D2R  REAL*8   1084
D2S  REAL*8   1092
D3F  REAL*8   1108
D3G  REAL*8   1140
D3P  REAL*8   1116
D3Q  REAL*8   1124
D3R  REAL*8   1148
D3S  REAL*8   1156
D4F  REAL*8   1172
D4G  REAL*8   1204
D4P  REAL*8   1180
D4Q  REAL*8   1188
D4R  REAL*8   1212
D4S  REAL*8   1220
ETA  REAL*8     0 /VECTOR/
ETAMAX REAL*8     0 *
F    REAL*8   1608 /VECTOR/
FNC  SUB/FUN PARM
G    REAL*8   6432 /VECTOR/
GNC  SUB/FUN PARM
H    REAL      4 *
I    INTEGER*4 976
ITRUBL INTEGER*4 0 /COMMQQ/
F    REAL*8   3216 /VECTOR/
FQ   REAL*8     8 *
Q    REAL*8   4824 /VECTOR/
QQ   REAL*8    12 *
R    REAL*8   8040 /VECTOR/
S    REAL*8   9648 /VECTOR/
SU   REAL*8    16 *
YF   REAL*8    20 *
YG   REAL*8    24 *
ZETA REAL*8     0 /BOUND /

```

333

Name	Type	Size	Class
BOUND		8	COMMON
COMMQQ		4	COMMON
CONTHS			PROGRAM
FF	REAL*8		FUNCTION
FG	REAL*8		FUNCTION
RUNGE			SUBROUTINE
VECTOR		11256	COMMON

Pass One No Errors Detected
333 Source Lines

APPENDIX D

SAMPLE OUTPUT FOR NON-SIMILAR BOUNDARY
LAYER PROBLEM

FREESTREAM VELOCITY = 1.00 VELOCITY AT THE WALL = .00
 INJECTION PARAMETER (ZETA) = .000

ETA	F	F*	F**
.00	.0000000	.0000000	.4696005
.20	.0093915	.0939054	.4693066
.40	.0375493	.1876054	.4672547
.60	.0843858	.2805758	.4617350
.80	.1496748	.3719637	.4511905
1.00	.2329905	.4606331	.4343796
1.20	.3336580	.5452471	.4105658
1.40	.4507243	.6243869	.3796926
1.60	.5829572	.6967002	.3424874
1.80	.7288733	.7610580	.3004456
2.00	.8867980	.8166953	.2556694
2.20	1.0549485	.8633049	.2105801
2.40	1.2315292	.9010661	.1675605
2.60	1.4148259	.9306019	.1286130
2.80	1.6032855	.9528761	.0951137
3.00	1.7955700	.9690552	.0677107
3.20	1.9905832	.9803655	.0463708
3.40	2.1874697	.9879710	.0305358
3.60	2.3855930	.9928884	.0193293
3.80	2.5845015	.9959448	.0117593
4.00	2.7838893	.9977706	.0068746
4.20	2.9835583	.9988189	.0038618
4.40	3.1833859	.9993972	.0020845
4.60	3.3832993	.9997038	.0010810
4.80	3.5832575	.9998601	.0005387
5.00	3.7832381	.9999365	.0002579
5.20	3.9832295	.9999725	.0001187
5.40	4.1832258	.9999888	.0000524
5.60	4.3832244	.9999959	.0000223
5.80	4.5832239	.9999988	.0000091
6.00	4.7832238	1.0000000	.0000036

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